



International Course on Architectural Design Second Level Degree Course

ARCHITECTURE and ENVIRONMENT LAB | ENVIRONMENTAL DESIGN

Computational Materiality for Sustainable Architectures and Comprehensive Skins

MODELING HUMAN BEHAVIORS AND COMPUTING COMFORT CONDITIONS

Prof. Arch. Giuseppe Ridolfi, PhD

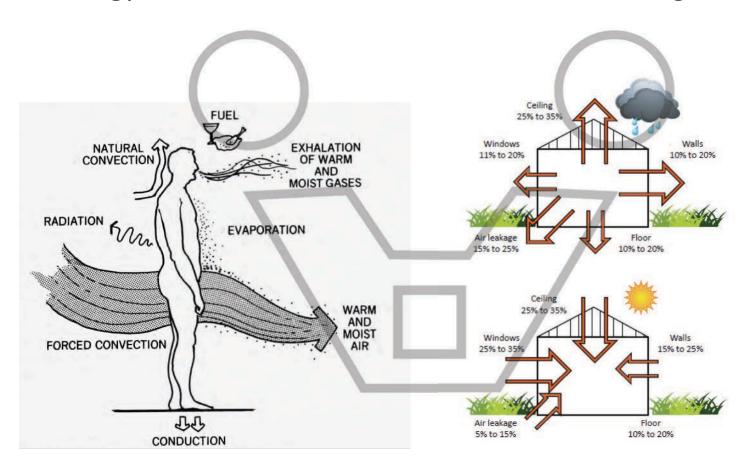
LECTURE #03

MODELING HUMAN BEHAVIORS AND COMPUTING COMFORT CONDITIONS

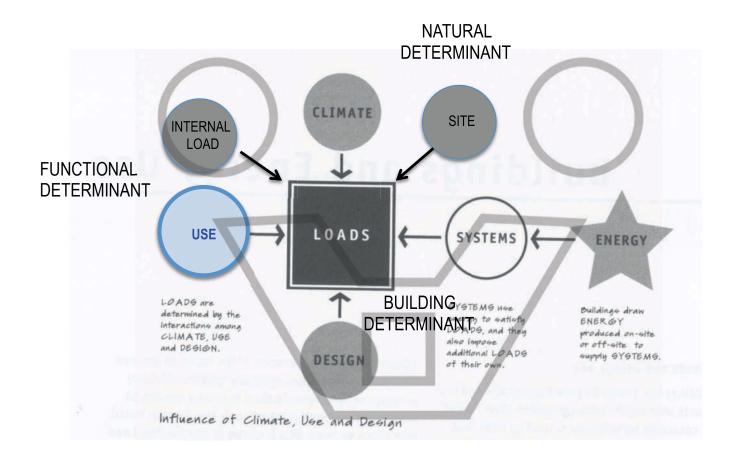
ENVIRONMENTAL ELEMENTS THAT AFFECT PEOPLE'S COMFORT



Energy mediator devices = Human skin & Building skin



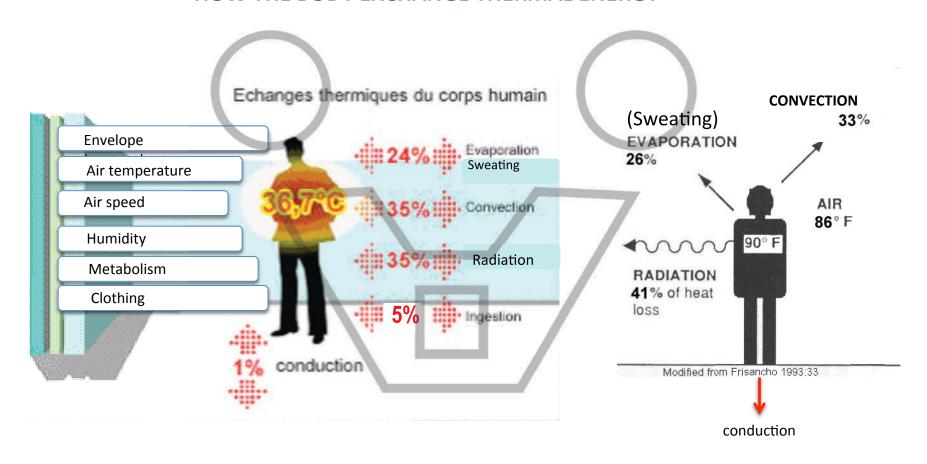






Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

HOW THE BODY EXCHANGE THERMAL ENERGY





Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

HOW THE BODY EXCHANGE THERMAL ENERGY

Environmental Temperature

Thermal comfort exists when a body's heat loss equals its heat gain or *vice versa*.

The body exchanges:

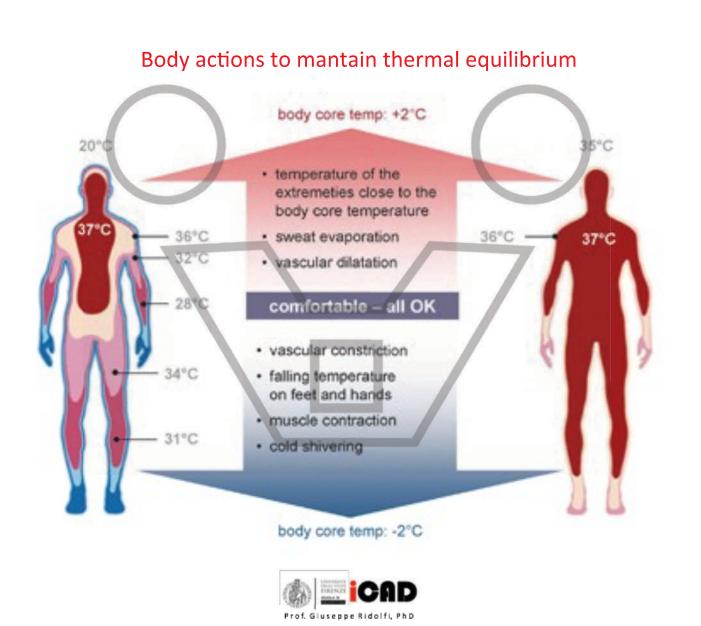
- ·62% of this heat via radiation,
- ·15% by evaporation,
- ·10% by convection,
- ·10% by respiration and
- \cdot 3% by conduction.

http://www2.ecospecifier.org/

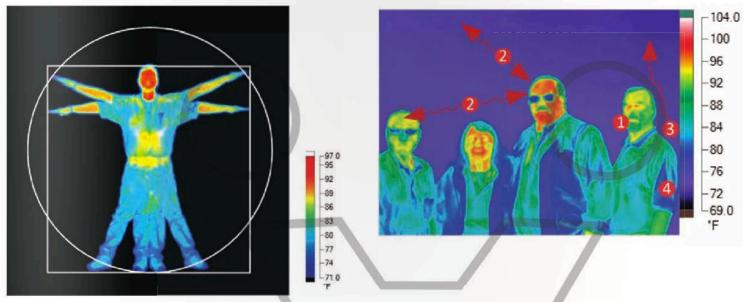
Relatively small changes in mean radiant temperature have a far greater effect than similar changes in air temperatures (Ballinger 1992). This gives rise to the importance of recognising the overall Environmental Temperature [T(env)], as opposed to just the dry bulb temperature.

T(env) = 2/3 Mean radiant surface temperature + 1/3 Air temperature

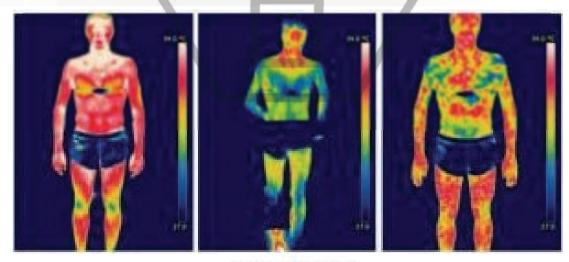
Thermal comfort = f (**TEMPERATURE**, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)



<u>Thermal comfort</u> = f (<u>TEMPERATURE</u>, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)



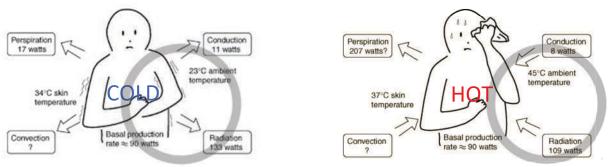
DIFFERENT HUMAN THERMAL ZONES

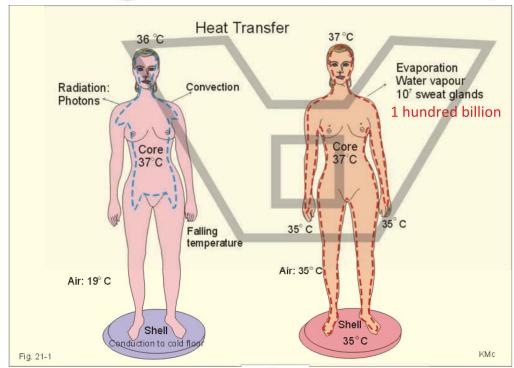




Thermal comfort = f (**TEMPERATURE**, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

Body actions to mantain thermal equilibrium







Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

EFFECT OF WIND ON TEMPERATURE (Apparent Temperature)

	Wind Speed	(mph)				
Temp (°C)	10	20	30	40	50	60
20	17	15	14	13	12	11
15	12	9	7	6	5	4
10	7	3	1	0	-2	-3
5	2	-3	-5	-7	-9	-10
0	-4	-9	-11	-14	-16	-17
-5	-9	-15	-18	-21	-23	-24
-10	-15	-21	-25	-28	-30	-32
-15	-21	-27	-32	-35	-37	-39
-20	-27	-33	-38	-42	-45	-47
		Significant	Severe	Extreme		

Wind chill equivalent temperatures from Steadman



<u>Thermal comfort</u> = f (TEMPERATURE, <u>WIND</u>, HUMIDITY, METABOLIC RATE, DRESSING RATE)

Appare	ent temperature (A	AT) as a Wind Chill - Temperature (°C)	- after Steadman 1994	
Apparent te	-2 -1 1 2 -2 -1 -1 -1 -1 -1 -1 -1 -1 -1 -1 -1 -1 -1	5 6 7 8 9 10 3 4 5 6 8 9 2 4 5 6 7 8 2 3 4 6 7 8 2 3 4 5 6 8 1 2 4 5 6 7 1 2 3 4 5 6 0 1 2 3 4 5 6 0 1 2 3 4 5 6 0 1 2 3 4 5 6 0 1 2 3 4 5 6 0 1 2 3 4 5 6 0 1 2 3 4 5 6 1 0 1 2 3 4 5 6 1 1 2 3 4 6 7 6 1 1 2 3 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	10	18 19 20 19 20 21 18 19 21 18 19 20 17 19 20 17 18 20 17 18 20 17 18 20 17 18 20 16 18 19 16 17 18 15 16 18 15 16 18 15 16 17 14 15 16 13 14 16 13 14 15 16 13 14 15 12 13 15 12 13 14 15 12 13 14 11 12 13 14 11 12 13 14 11 13 14 15 12 13



Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE) WIND CHILL- Siple e Passel del 1945 reviewed in 2001

		111		177				Air T	emper	ature	(Celsiu	IS)				HII)		
		0	-1	-2	-3	-4	-5	-10	-15	-20	-25	-30	-35	-40	-45	-50	-55	-60
	6	-2	-3	4	-5	-7	-8	-14	-19	-25	-31	-37	-42	-48	-54	-60	-65	-71
- 1	8	-3	-4	-5	-6	-7	-9	-14	-20	-26	-32	-38	-44	-50	-56	-61	-67	-73
- 1	10	-3	-5	-6	-7	-8	-9	-15	-21	-27	-33	-39	-45	-51	-57	-63	-69	-75
1	15	4	-6	-7	-8	-9	-11	-17	-23	-29	-35	-41	-48	-54	-60	-68	-72	-78
- 1	20	-5	-7	-8	-9	-10	-12	-18	-24	-30	-37	-43	-49	-56	-62	-68	-76	-81
- 1	25	-6	-7	-8	-10	-11	-12	-19	-25	-32	-38	-44	-51	-57	-64	-70	-77	-83
	30	-6	-8	-9	-10	-12	-13	-20	-26	-33	-39	46	-52	-69	-65	-72	-78	-85
(km/hr)	35	-7	-8	-10	-11	-12	-14	-20	-27	-33	-40	-47	-63	-60	-66	-73	-80	-86
2	40	-7	-9	-10	-11	-13	-14	-21	-27	-34	-41	-48	-54	-61	-68	-74	-81	-88
2	45	-8	-9	-10	-12	-13	-15	-21	-28	-35	-42	48	-55	-62	-69	-75	-82	-89
=	50	-8	-10	-11	-12	-14	-15	-22	-29	-35	-42	-49	-56	-63	-69	-76	-83	-90
ě	55	-8	-10	-11	-13	-14	-15	-22	-29	-36	-43	-50	-57	-63	-70	-77	-84	-91
Speed	60	-9	-10	-12	-13	-14	-16	-23	-30	/-36	-43	-50	-57	-64	-71	-78	-85	-92
S	65	-9	-10	-12	-13	-15	-16	-23	-30	-37	-44	-51	-58	-65	-72	-79	-86	-93
Wind	70	-9	-11	-12	-14	-15	-16	-23	-30	-37	-44	-51	-58	-65	-72	-80	-87	-94
5	75	-10	-11	-12	-14	-15	-17	-24	-31	-38	-45	-52	-59	-66	-73	-80	-87	-94
>	80	-10	-11	-13	-14	-15	-17	-24	-31	-38	-45	-82	-60	-67	-74	-81	-88	-95
ı	85	-10	-11	-13	-14	-16	-17	-24	-31	-39	-46	-63	-60	-67	-74	-81	-89	-96
ı	90	-10	-12	-13	-15	-16	-17	-25	-32	-39	46	-53	-61	-68	-75	-82	-89	-96
	95	-10	-12	-13	-15	-16	-18	-25	-32	-39	-47	-54	-61	-68	-75	-83	-90	-97
1	100	-11	-12	-14	-15	-16	-18	-25	-32	-40	-47	-54	-61	-69	-76	-83	-90	-98
ŀ	105	-11	-12	-14	-15	-17	-18	-25	-33	-40	47	-55	-62	-69	-76	-84	-91	-98
1	110	-11	-12	-14	-15	-17	-18	-26	-33	-40	-48	-65	-62	-70	-77	-84	-91	-99
	1,10	0 to			-10 to	25 Mode		-25 to - 4				59 Extre	me		very Ex	_		

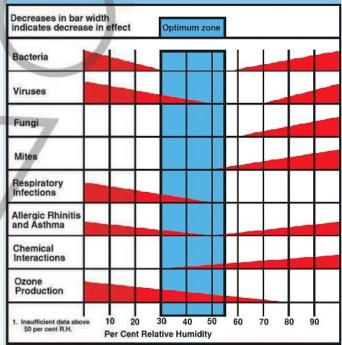


Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

HUMIDEX- PERCEIVED TEMPERATURE AND DISCOMFORT INDEX (range 20°C-55°C)

	25%	30%	35%	40%	45%	50%	55%	60%	65%	70%	75%	80%	85%	90%	95%	100%
42°	48	50	52	55	57	59	62	64	66	68	71	73	75	77	80	82
41°	46	48	51	53	55	57	59	61	64	66	88	70	72	74	76	79
40°	45	47	49	51	53	55	57	59	61	63	65	87	69	71	73	75
39°	43	45	47	49	51	53	55	57	59	61	83	85	88	68	70	72
38°	42	44	45	47	49	51	53	55	56	58	60	82	84	68	67	69
37°	40	42	44	45	47	49	51	52	54	56	58	59	61	63	65	66
36°	39	40	42	44	45	47	49	50	52	54	55	57	59	60	62	63
35°	37	39	40	42	44	45	47	48	50	51	53	54	58	58	59	61
34°	36	37	39	40	42	43	45	46	48	49	51	52	54	55	57	58
33°	34	36	37	39	40	41	43	44	46	47	48	50	51	53	54	- 55
32°	33	34	36	37	38	40	41	42	44	45	46	48	49	50	52	53
31°	32	33	34	35	37	38	39	40	42	43	44	45	47	48	49	50
30°	30	32	33	34	35	36	37	39	40	41	42	43	45	46	47	48
29°	29	30	31	32	33	35	36	37	38	39	40	41	42	43	45	46
28°	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43
27°	27	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41
26°	26	26	27	28	29	30	31	32	33	34	34	35	36	37	38	39
25°	25	25	26	27	27	28	29	30	31	32	33	34	34	35	36	37
24°	24	24	24	25	26	27	28	28	29	30	31	32	33	33	34	35
23°	23	23	23	24	25	25	26	27	28	28	29	30	31	32	32	33
22°	22	22	22	22	23	24	25	25	26	27	27	28	29	30	30	31

OPTIMUM INDOOR RELATIVE HUMIDITY & AIR QUALITY GUIDE



Fino a 29 C°

Nessun disagio

Da 30 a 34 C° Sensazione di disagio

Da 35 a 39 C° Intenso disagio. Prudenza: limitare le attività fisiche più pesanti

Da 40 a 45 C° Forte sensazione di malessere. Pericolo: evitare gli sforzi

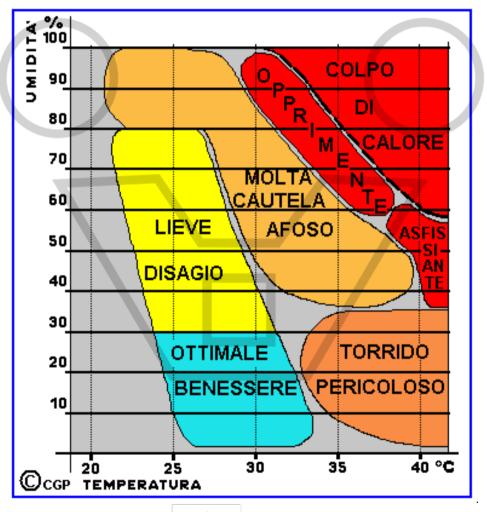
Da 46 a 53 Cº Pericolo grave: interrompere tutte le attività fisiche

Pericolo di morte: colpo di calore imminente



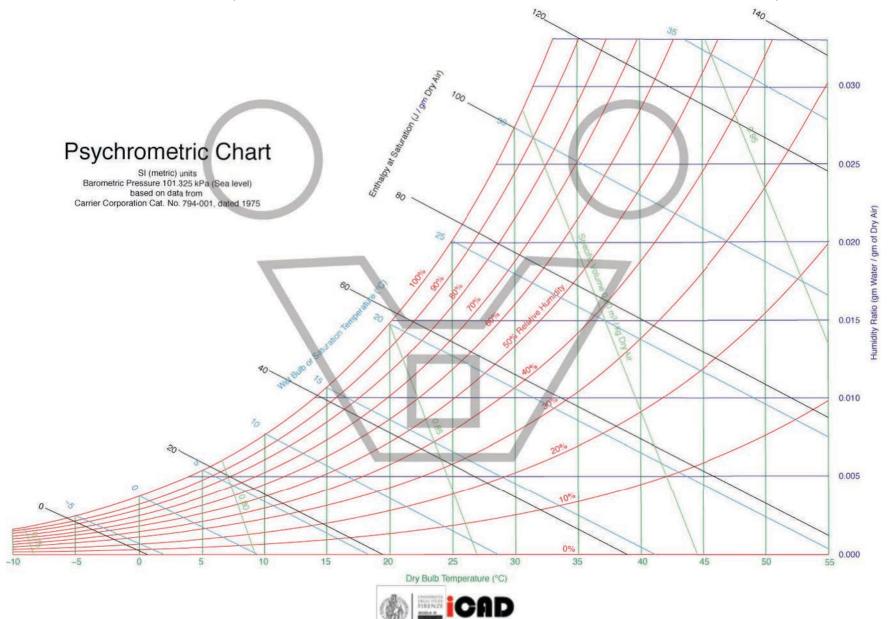
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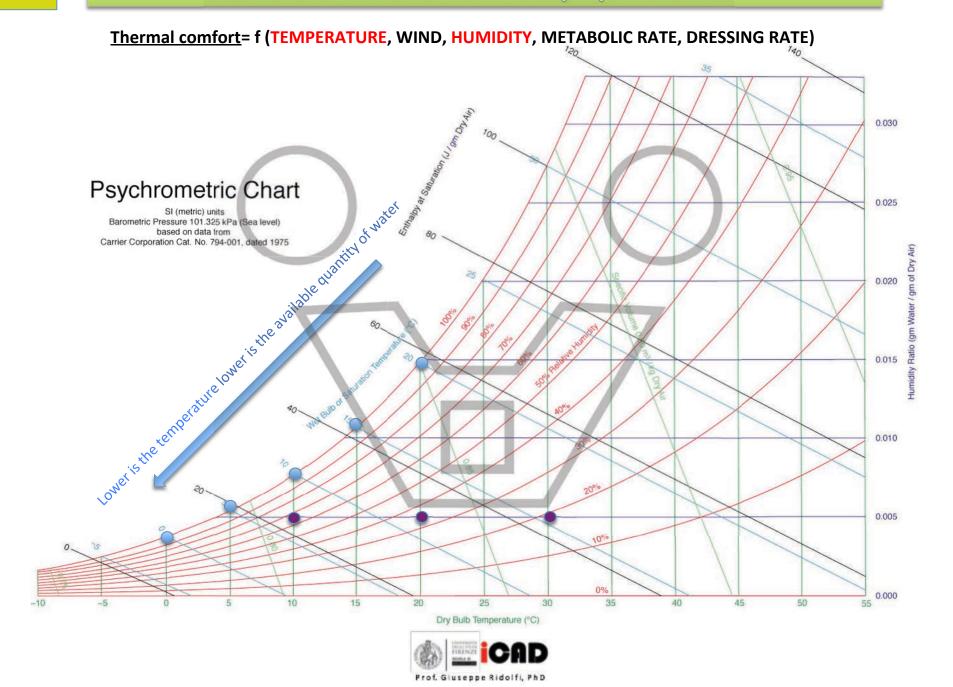
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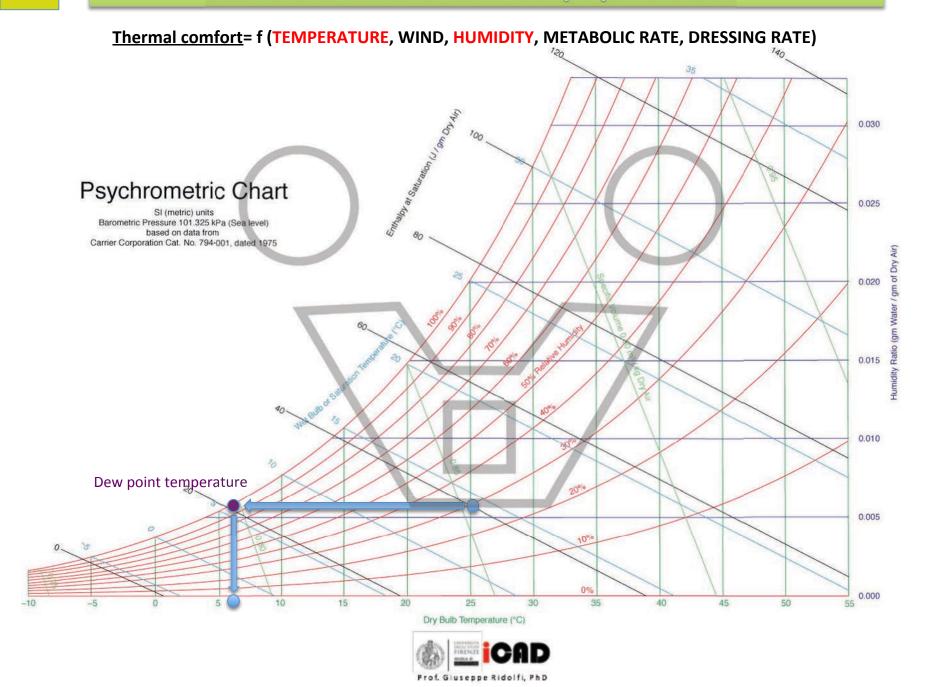


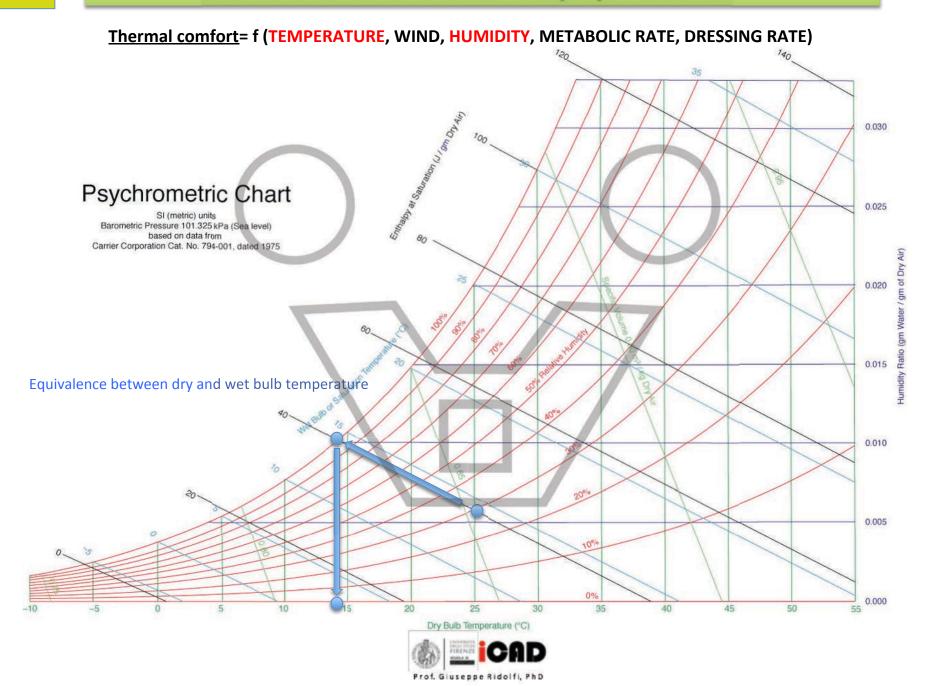


Thermal comfort = f (**TEMPERATURE**, WIND, **HUMIDITY**, METABOLIC RATE, DRESSING RATE)









Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

EFFECTIVE TEMPERATURE & THERMAL COMFORT ZONE

	25%	30%	35%	40%	45%	50%	55%	60%	65%	70%	75%	80%	85%	90%	95%	100%
2°	48	50	52	55	57		62	64	66	68	71	73	75	77		82
и°	46	48	51	53	55	57	59	61	64	66	88	70	72	74	76	79
10°	45	47	49	51	53	55	57	59	61	63	65	87	69	71	73	75
9°	43	45	47	49	51	53	55	57	59	61	83	65	68	68	70	72
8°	42	44	45	47	49	51	53	/55	56	58	60	62	64	66	67	69
7°	40	42	44	45	47	49	51	52	54	56	58	59	61	63	65	66
20	39	40	42	44	45	6	49	50	52	54	55	57	59	60	62	63
5°	37 -	-39	40	42	44	45	47	48	50	51	53	54	58	58	59	61
	36	37	39.	40	42	V	45	46	48	49	51	52	54	55	57	58
3°	34	36	37	39 -	-40	41	43	44	46	47	48	50	51	53	54	- 55
2°	33	34	36	37	38	- 40.	41	42	44	45	46	48	49	50	52	53
10	32	33	34	35	37	38	39 -	_40	42	43	44	45	47	48	49	50
10°	30	32	33	34	35	36	37	39	- 40	41	42	43	45	46	47	48
9°	29 -	30	31	32	33	35	36	37	38	39 -	40	41	42	43	45	46
80	28	29	-30.	31	32	33	34	35	36	37	38	- 39	40	41	42	43
? 7°	27	27	28	29	_30	31	32	33	34	35	36	37	-38~	39	40	41
6°	26	26	27	28	29	30-	31	32	33	34	34	35	36	37	- 38	39
.5°	25	25	26	27	27	28	29	-30_	31	32	33	34	34	35	36	-37
40	24	24	24	25	26	27	28	28	29	- 30	31	32	33	33	34	35
3°	23	23	23	24	25	25	26	27	28	28	29 -	_30	31	32	32	33
2°	22	22	22	22	23	24	25	25	26	27	27	28	- 29	30	30	31

Fino a 29 C° Nessun disaglo

Da 30 a 34 C° Sensazione di disaglo

Da 35 a 39 C° Intenso disaglo. Prudenza: limitare le attività fisiche più pesanti

Forte sensazione di malessere. Pericolo: evitare gli sforzi

Da 46 a 53 C° Pericolo grave: interrompere tutte le attività fisiche

Olime 54 C° Pericolo di morte: colpo di calore imminente

The humidex is a Canadian creation first used in 1965



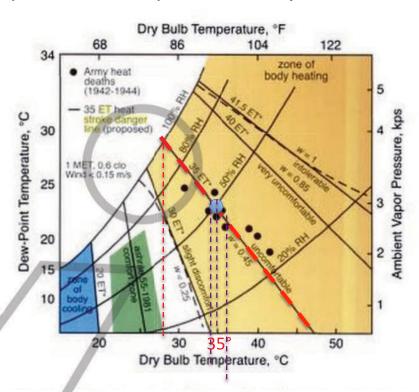
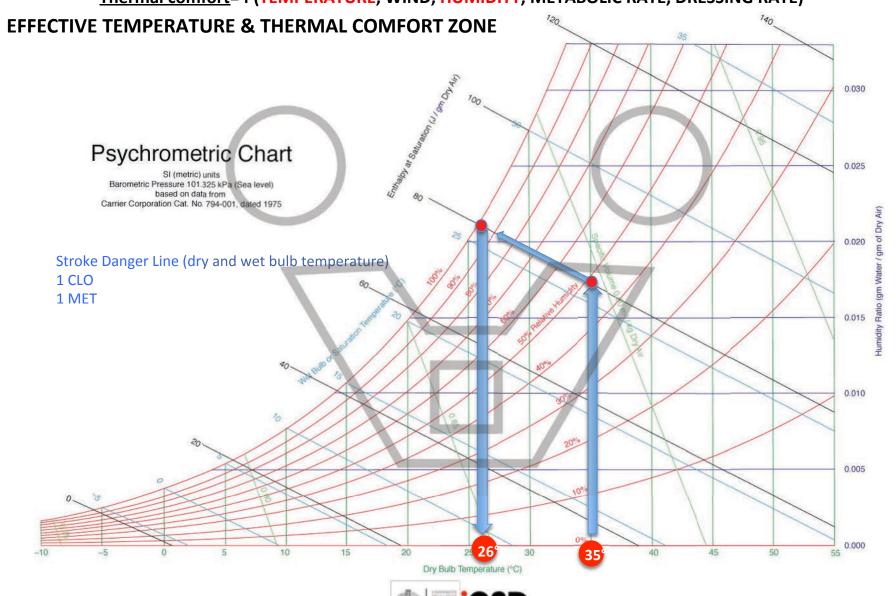


Fig. 8-8. Nomogram for the "new," effective temperature (ET*), including data points for US Army heat deaths provided by Hardy. ashrae: American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc; clo: unit of clothing insulation. 1 clo = 0.155 m² K/W; MET: unit of metabolism, 1 MET = 58.15 W/m². RH: relative humidity. Illustration: Adapted with permission from ASHRAE. Copyright 2005 © American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc (www.ashrae.org). Reprinted with permission from the 2005 ASHRAE Handbook—Fundamentals. (This text may not be copied nor distributed in either paper or digital form without ASHRAE's permission.) Data source: Hardy JD. Thermal comfort and health. ASHRAE J. 1971;13:43.

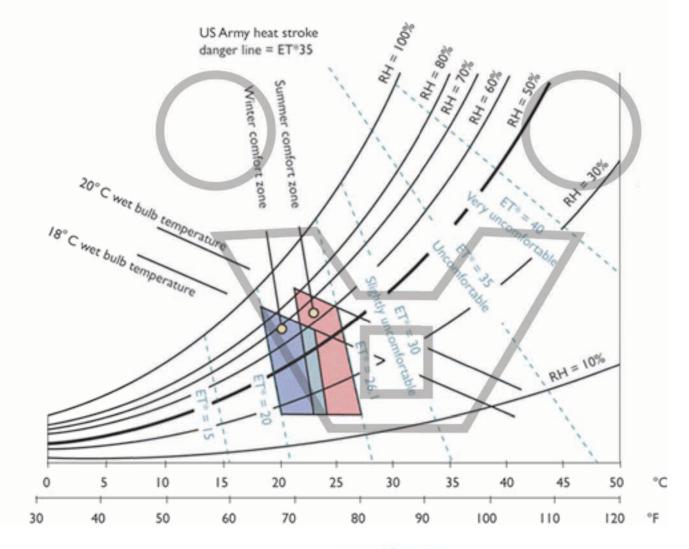
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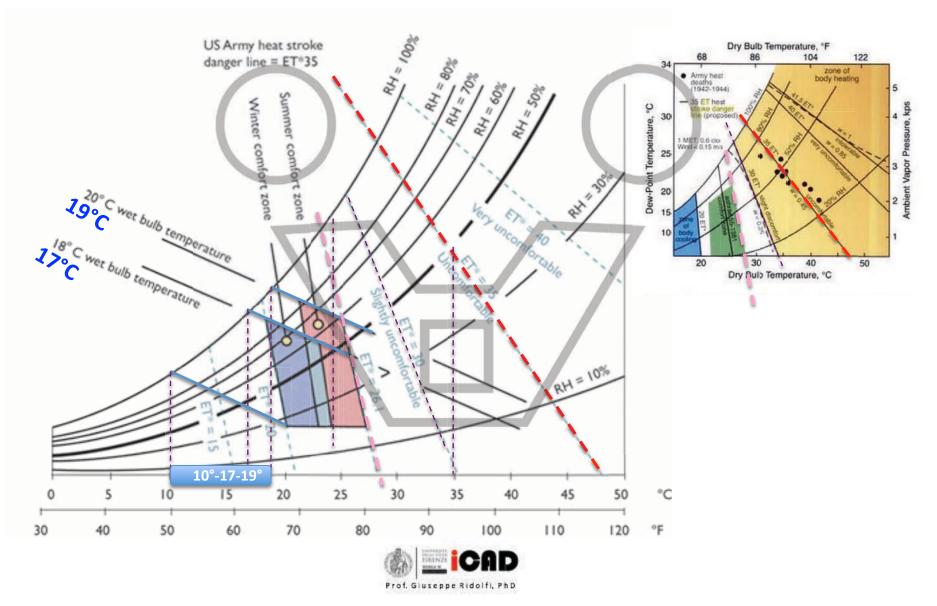
EFFECTIVE TEMPERATURE & THERMAL COMFORT ZONE





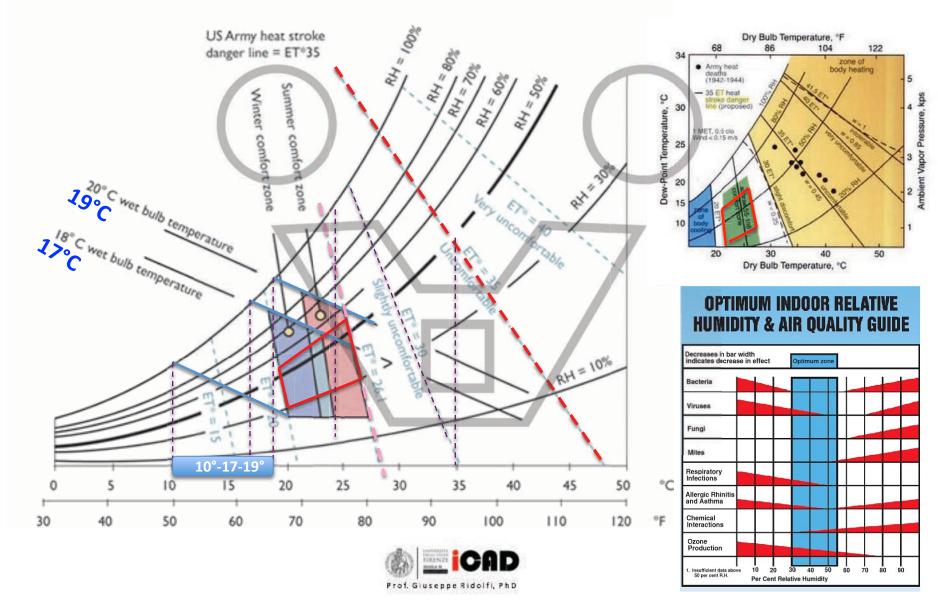
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EFFECTIVE TEMPERATURE & THERMAL COMFORT ZONE _ Using wet bulb temperature



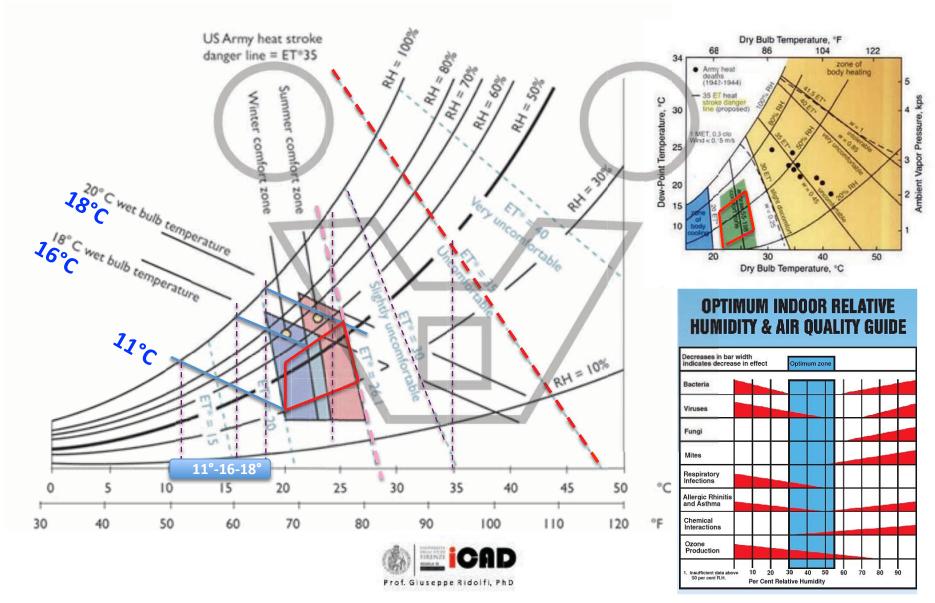
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EFFECTIVE TEMPERATURE & THERMAL COMFORT ZONE _ Using wet bulb temperature

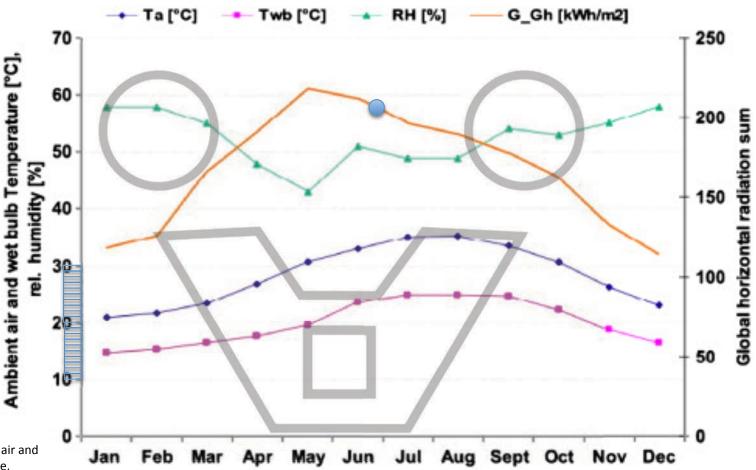


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EFFECTIVE TEMPERATURE & THERMAL COMFORT ZONE _ Using wet bulb temperature



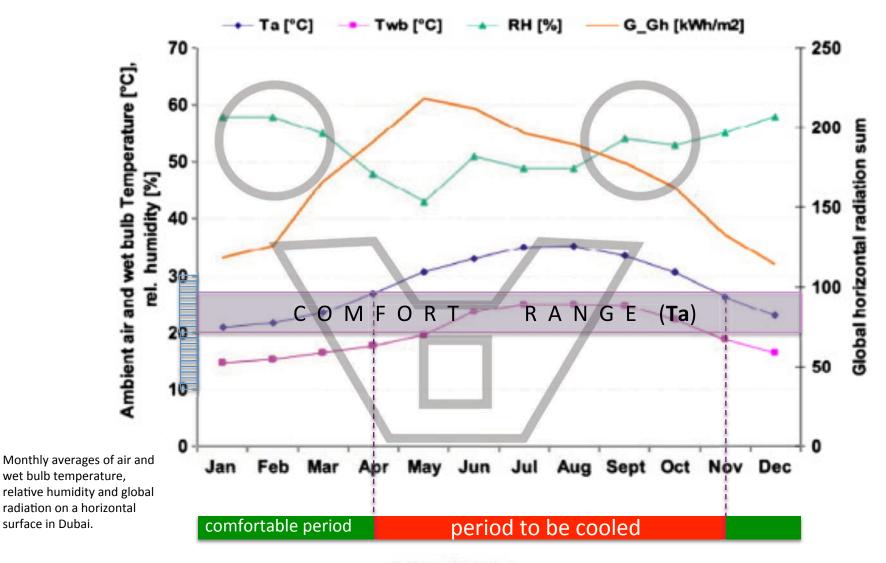
IDENTIFYING THE COMFORT PERIOD



Monthly averages of air and wet bulb temperature, relative humidity and global radiation on a horizontal surface in Dubai.

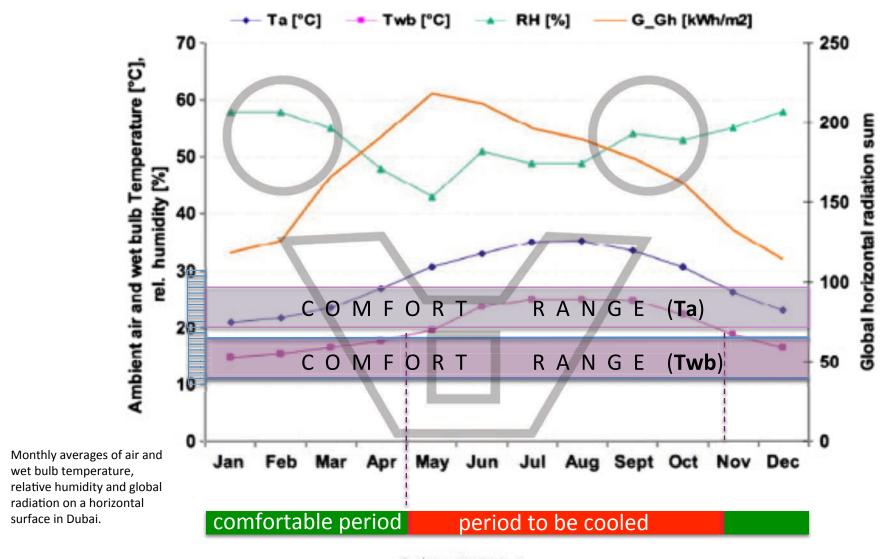


IDENTIFYING THE COMFORT PERIOD

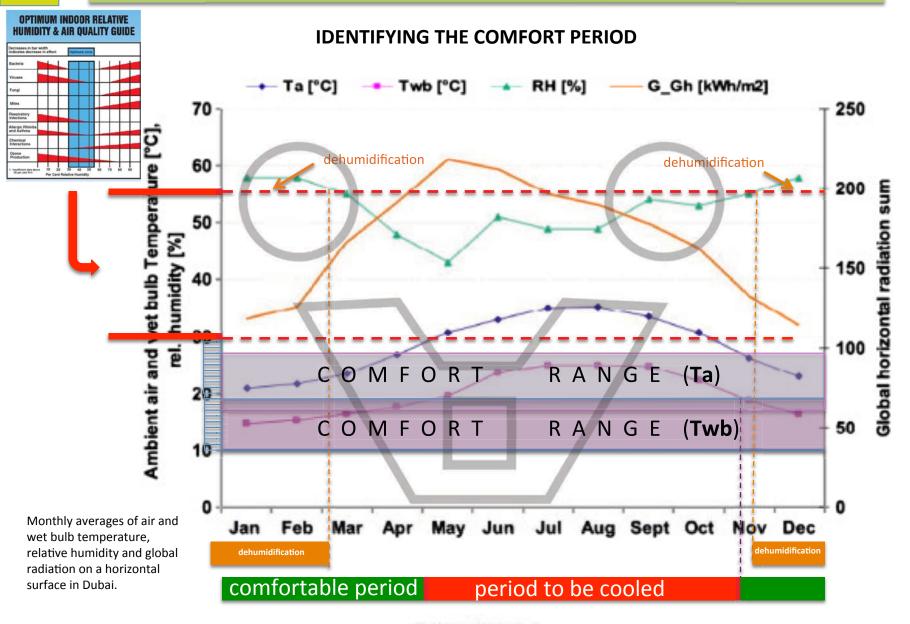




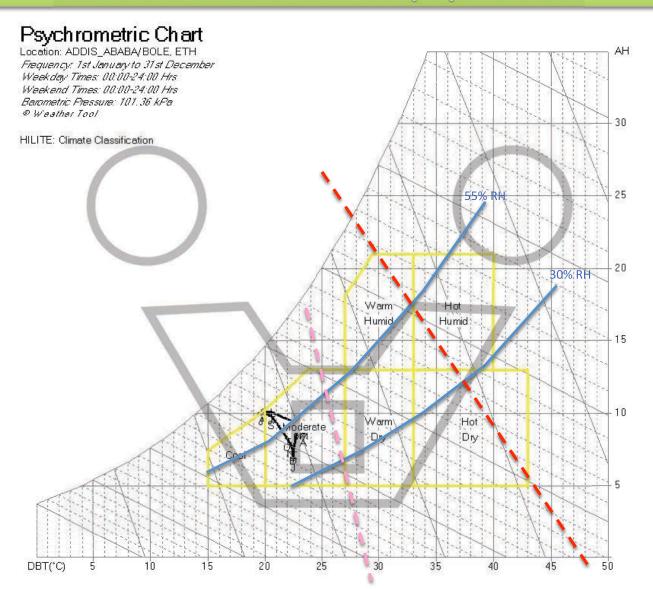
IDENTIFYING THE COMFORT PERIOD





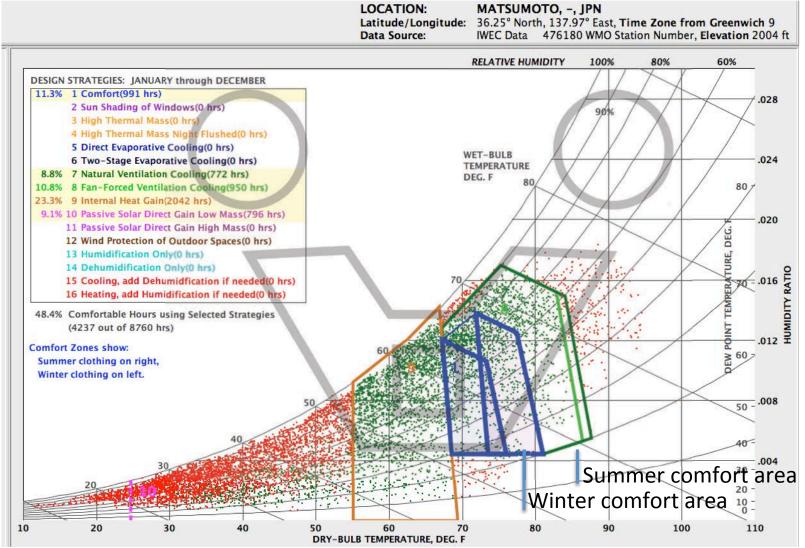








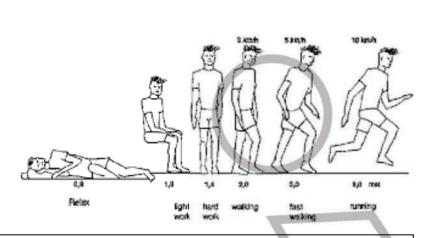
IDENTIFYING THE COMFORT PERIOD



Ecotec output from climate-consultant http://www.energy-design-tools.aud.ucla.edu/



<u>Thermal comfort</u> = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)



 $1met = 50kcal/hm^2 = 0.05815Kw/hsqm$ 1 kcal = 4.190 J = 1.164 Wh.

1 BTU/h ft2= 5,6783 w/h mq 1 w/h mq= 0,17610904672173 BTU/h ft2 http://www.the-engineering-page.com/conv/u.html

1 BTU = 252 cal

1 BTU = 1,055056 kJ

1 W = 3,412 BTU/h = 1J/s

1 Kcal= 0,00116 Kw/h

http://www.convertworld.com

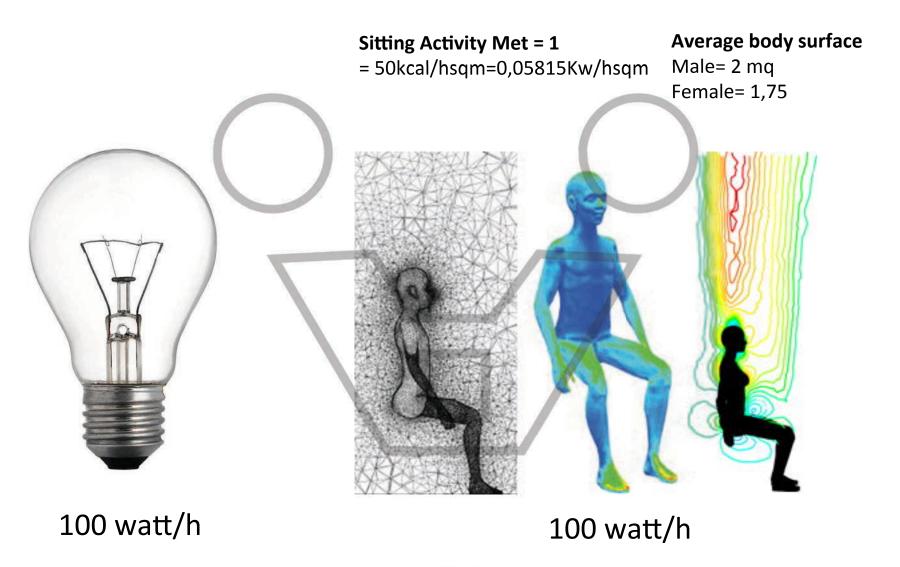


Table 4	Typical Metabolic Heat Generation for
	Various Activities

(a)	Btu/h-ft ²	met*
Resting	1944	9763
Sleeping	13	0.7
Reclining	15	0.8
Scated, quiet	18	1.0
Standing, relaxed	22	1.2
Walking (on level surface)		
2.9 (ps (2 mph)	3.7	2.0
4.4 fps (3 mph)	48	2.6
5.9 fps (4 mph)	70	3.8
Office Activities		
Reading, seated	18	1.0
Writing	18	1.0
Typing	20	1.1
Filing, seated	22	1.2
Filing, standing	26	1.4
Walking about	31	1.7
Lifting/packing	39	2.1
Driving/Flying		
Car	18 to 37	1.0 to 2.0
Aircraft, routing	22	1.2
Aircraft, instrument landing	3.3	1.8
Aircraft, combat	44	2.4
Heavy vehicle	59	3.2
Miscellaneous Occupational Activitie	s	
Cooking	29 to 37	1.6 to 2.0
Housecleaning	37 to 63	2.0 to 3.4
Seated, heavy limb movement	41	2.2
Machine work	72	
sawing (table saw)	33	1.8
light (electrical industry)	37 to 44	2.0 to 2.4
heavy	74	4.0
Handling 110 lb bags	74	4.0
Pick and shovel work	74 to 88	4.0 to 4.8
Miscellaneous Leisure Activities		
Dancing, social	44 to 81	2.4 to 4.4
Calisthenics/exercise	55 to 74	3.0 to 4.0
Tennis, singles	66 to 74	3.6 to 4.0
Basketball	90 to 140	5.0 to 7.6
Wrestling, competitive	130 to 160	7.0 to 8.7

Sources: Compiled from various sources. For additional information, see Buskirk (1960), Passmore and Dumin (1967), and Webb (1964).

Thermal comfort = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)



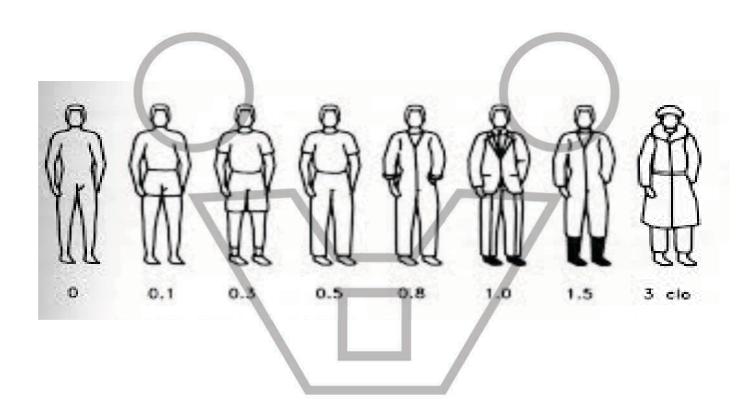




Converting the excess heat generated by the Stockholm Central Station's 2500 daily users to hot water and pump it to the nearby Kungsbrohuset office block.

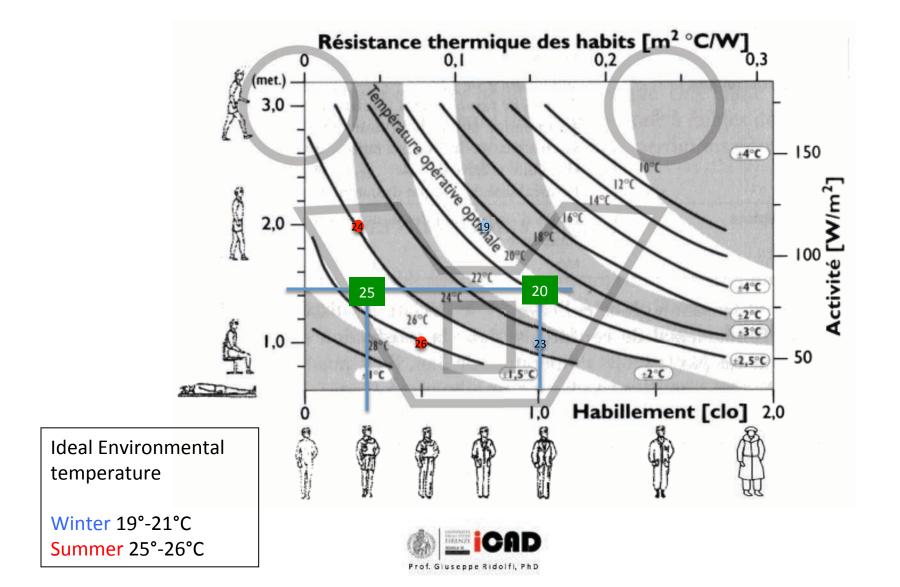


<u>Thermal comfort</u> = f (TEMPERATURE, WIND, HUMIDITY, METABOLIC RATE, DRESSING RATE)

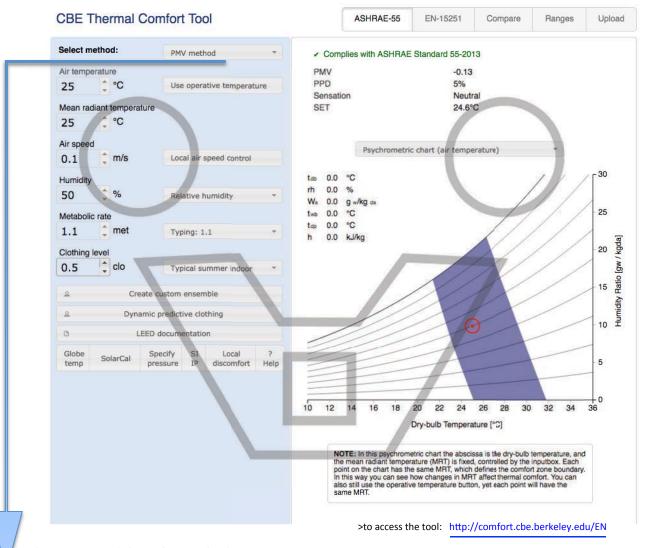




THERMAL COMFORT ZONE



THERMAL COMFORT ZONE



The **PMV** model can be applied to air conditioned buildings,
The **Adaptive** model can be generally applied only to buildings where no mechanical systems have been installed



MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

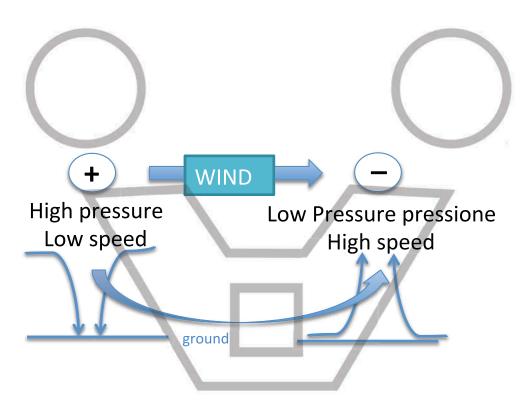
WIND & VENTILATION

Air movement: WIND & CROSS VENTILATION

- Temperature balance
- Quality of indoor air [Minimal air change = 0,5 vol/h]
- Facilitate human transpiration

Air movement: WIND & CROSS VENTILATION

f [work done f(pressure), cinetic energy f(velocity)+potential gravitational energy f(high)]

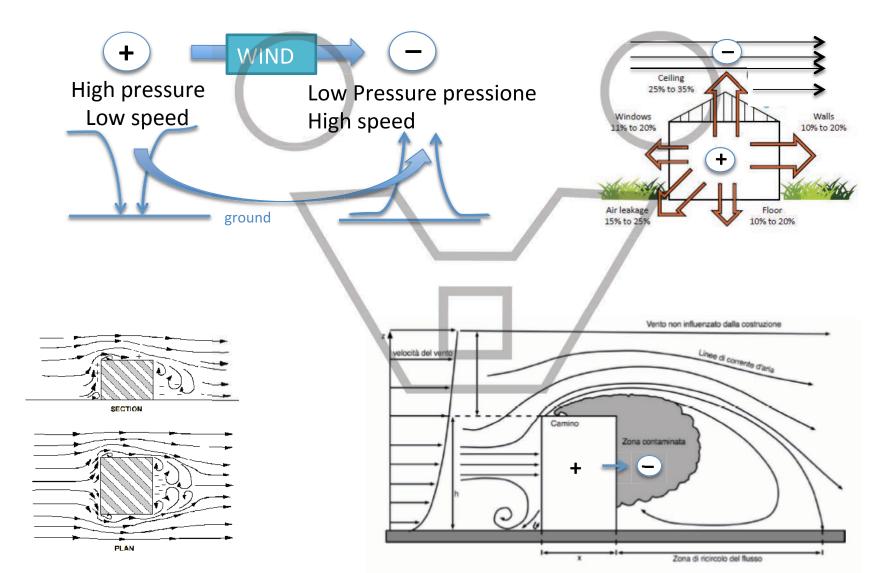


BERNOULLI PRINCIPLE= higher speed lower pressure

Air movement: WIND & CROSS VENTILATION

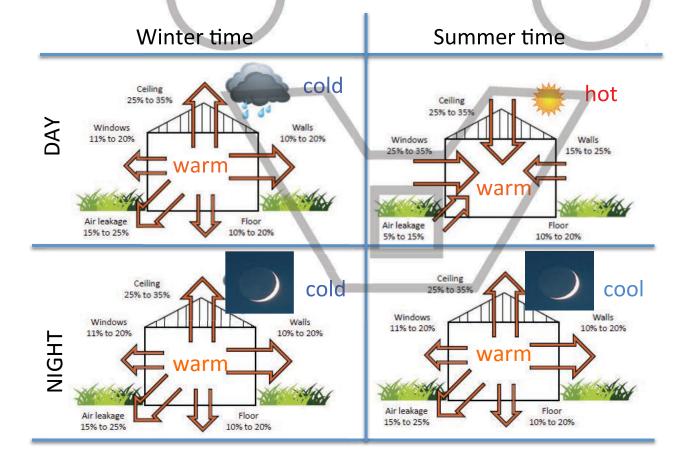
f [work done f(pressure), cinetic energy f(velocity)+potential gravitational energy f(high)]

Bernoulli principle= higher speed lower pressure

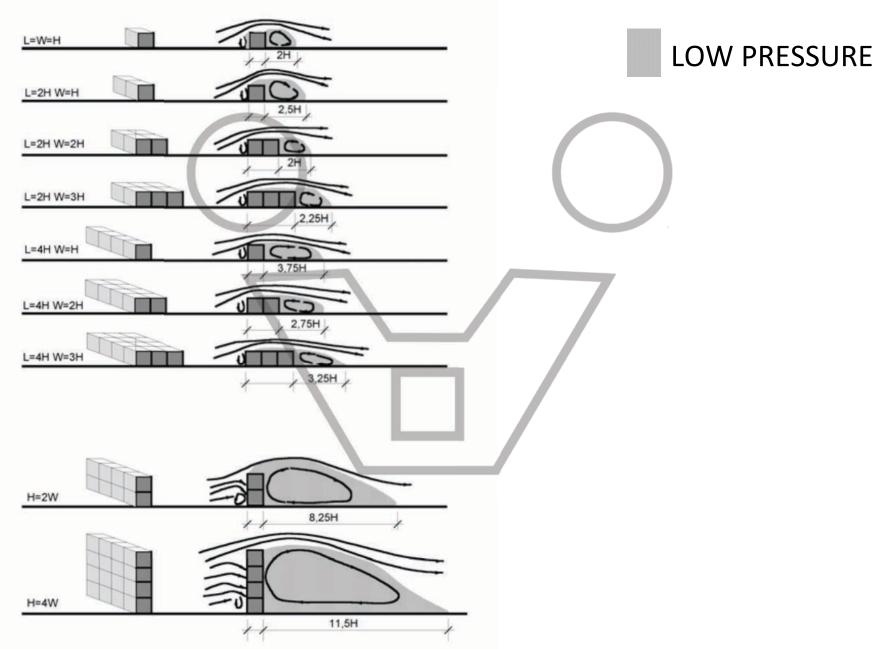


Air movement: WIND & CROSS VENTILATION

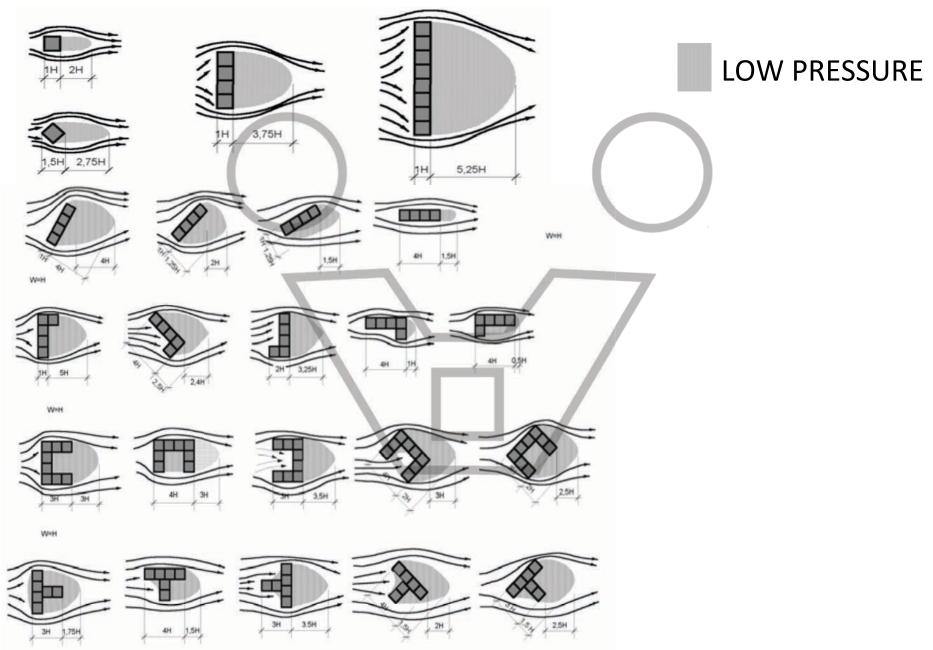
<u>From higher temperature</u> (lower density, lower weight, higher pressure) <u>To lower temperature</u> (higher density, higher weight, lower pressure)



Air movement: WIND ANALYSIS



Air movement: WIND ANALYSIS



Air movement: WIND ANALYSIS

HOW TO VISUALIZE WINDS – Airflow Modeling

Understanding the air flow and distribution patterns for buildings.

The building form and shape can affect how air flows through the building and across neighboring developments into the building.

This is an important consideration for natural ventilation and can significantly reduce costs of air-conditioning provisions. There are Computational Fluid Dynamics (CFD) tools available that can help simulate the air-flow patterns within built-spaces as well as for whole building estates

Basic software tool:

Flow Design http://www.autodesk.com/education/free-software/flow-design (student version available)

Other popular software tools:

Fluent by Ansys: http://www.ansys.com/. (student version available)

FloVent from Mentor Graphics: http://www.mentor.com/.

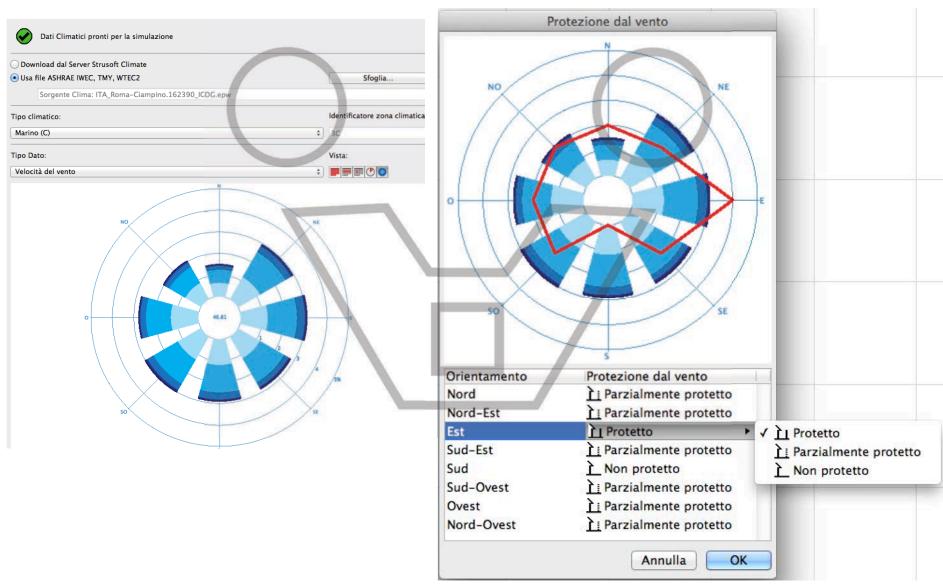
Comsol Multiphysics modeling software: https://www.comsol.com/.

<u>References</u>

AIA (The American Institute of Architects) (2012) An Architect's guide to integrating energy modeling in the design process ERI@N (Energy Research Institute @ NTU) (2013) Nanyang Technological University (NTU), Singapore NREL (2009) A handbook for planning and conducting charrettes for high-performance projects, National Renewable Energy Laboratory (NREL), Sept 2009

Air movement: WIND ANALYSIS

- 1- determine the coldest and the hottest seasonal period and hours
- 2- for that periods find the most frequent wind directions



Air movement: WIND ANALYSIS

- 3- define wind speed for the hottest and coldest periods
- 4- reduce the speed according to altitude and roughness of the site

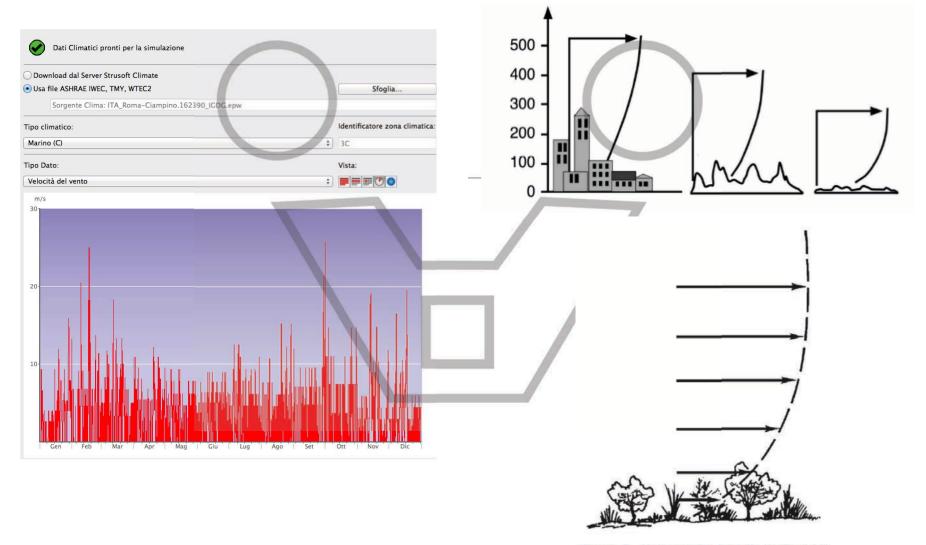
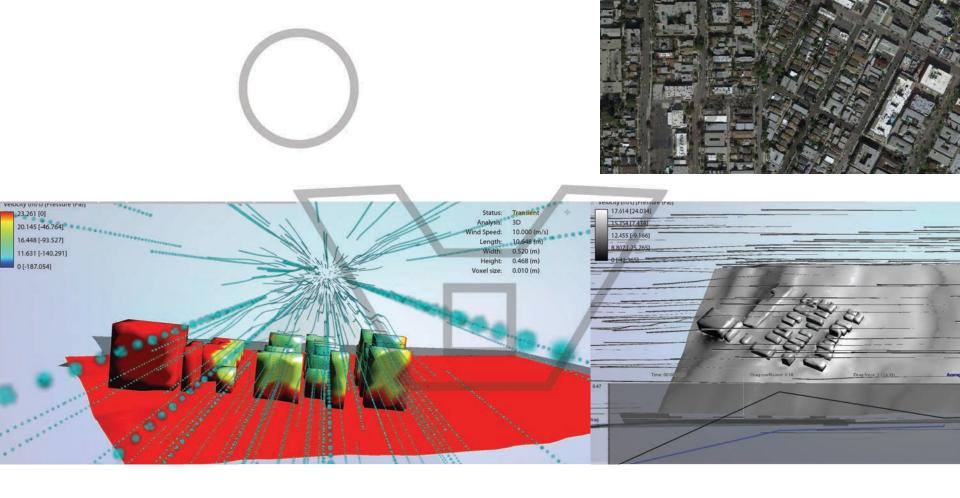


Figure 1—General wind velocity profile near surface (from Rothermel 1983).

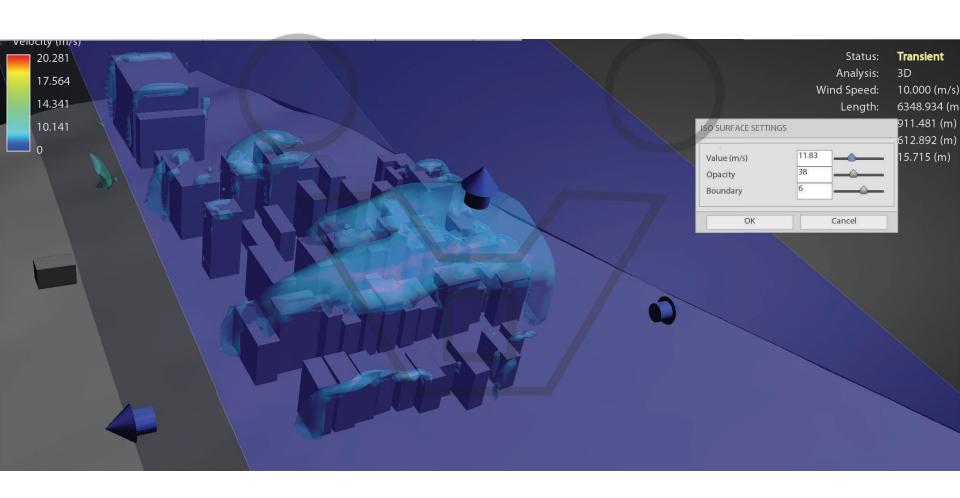
Air movement: WIND ANALYSIS

5- Orient the model according to the wind direction

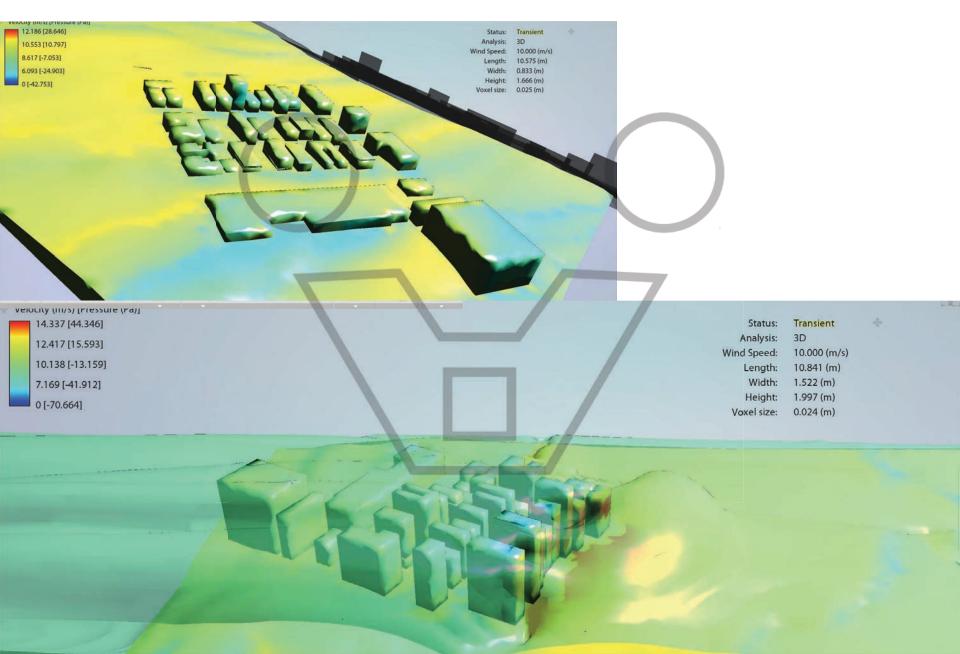


Air movement: WIND ANALYSIS

6- Set the wind velocity & analyze results (low, high pressure zones)

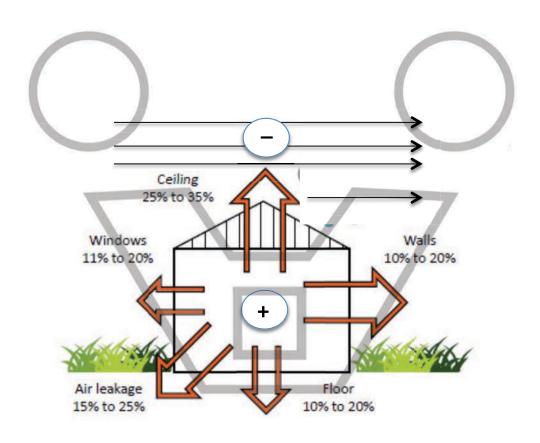


Air movement: WIND ANALYSIS



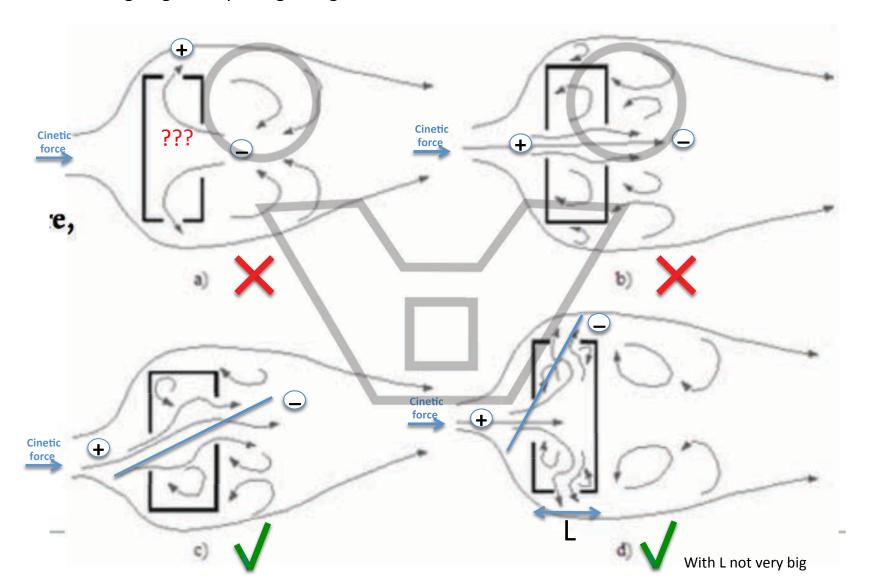
Air movement: WIND ANALYSIS

7- Design buildings according the wind pressure zones and cinetic forces



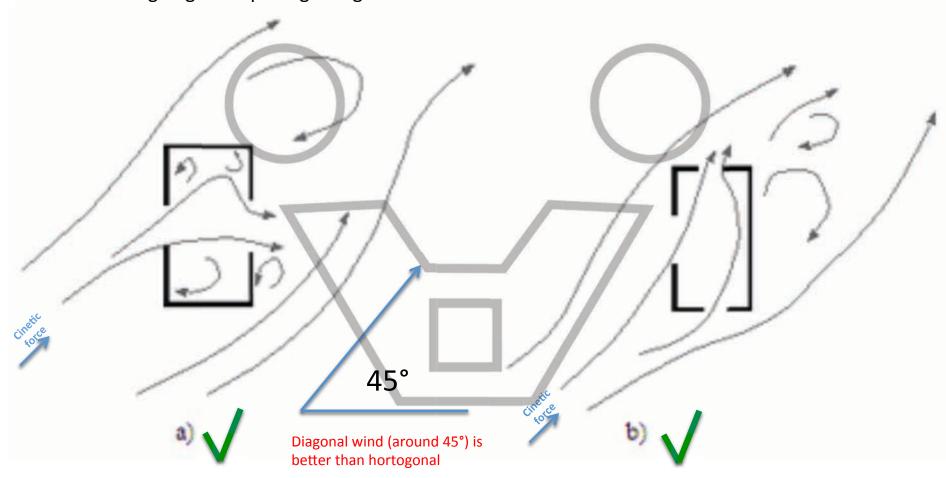
Air movement: WIND ANALYSIS

- 7- Design buildings according the wind pressure zones and cinetic forces
 - Designing skin openings for good cross ventilation



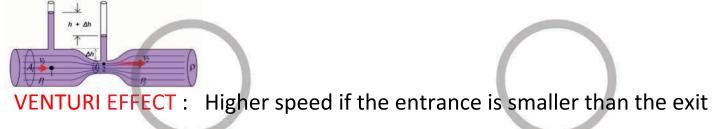
Air movement: WIND ANALYSIS

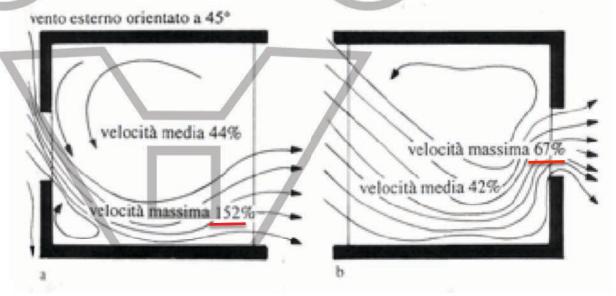
- 7- Design buildings according the wind pressure zones and cinetic forces
 - Designing skin openings for good cross ventilation



Air movement: WIND ANALYSIS

- 7- Design buildings according the wind pressure zones and cinetic forces
 - Designing skin openings for good cross ventilation





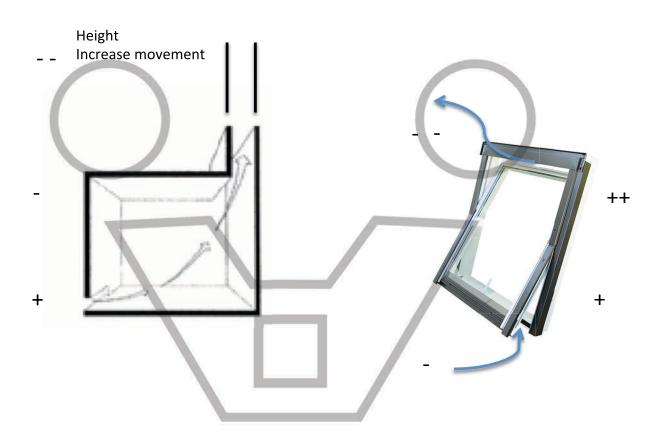
Pairing a large outlet with a small inlet increases incoming wind speed.

MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

STACK EFFECT & VENTILATION

Air movement: VERTICAL VENTILATION

f [work done f(pressure), cinetic energy f(velocity)+potential gravitational energy f(high)]

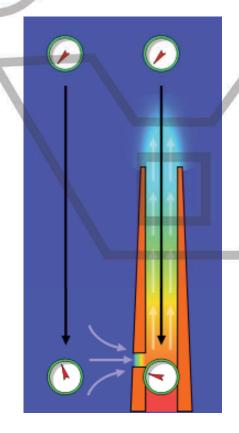


Air movement: VERTICAL VENTILATION

Stack Effect >>> BUOYANCY EFFECT

<u>From higher temperature</u> (lower density, lower weight, higher pressure) <u>To lower temperature</u> (higher density, higher weight, lower pressure)

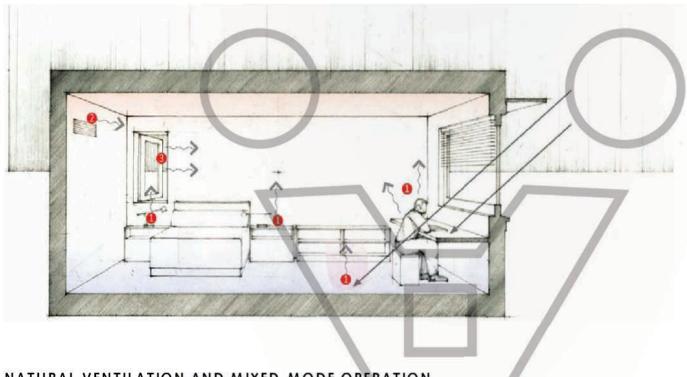
In addition, the stack effect use the difference in altitude (potential gravitational energy)



Typically, at night, wind speeds are slower, so ventilation strategies driven by wind is less effective.
Therefore, stack ventilation is an important strategy.

Air movement: VERTICAL VENTILATION

Multiple sources airflow



NATURAL VENTILATION AND MIXED-MODE OPERATION

Natural ventilation channels direct outside air to do one or more of the following: provide fresh air, provide cooling, or provide airflow to increase human comfort levels. The ability to use natural ventilation depends climate wise on peak interior temperatures, and the humidity solar radiation, and wind

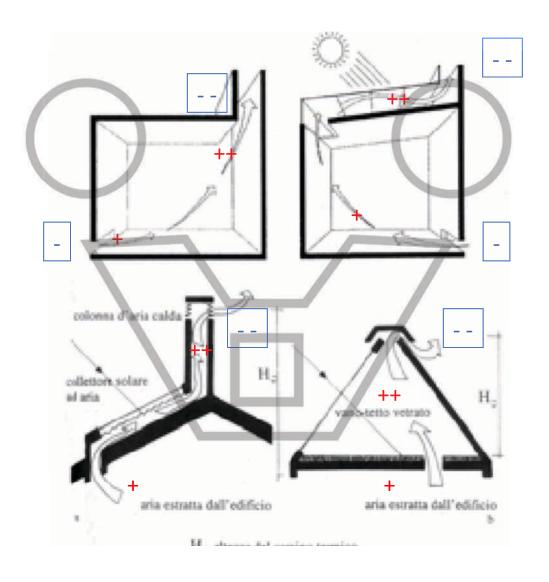
office and conference room spaces in plan and section. CFD was used to verify the potential for natural ventilation to provide adequate cooling even when outdoor windspeeds are low.

Source: Courtesy of KEMA Energy Analysts.

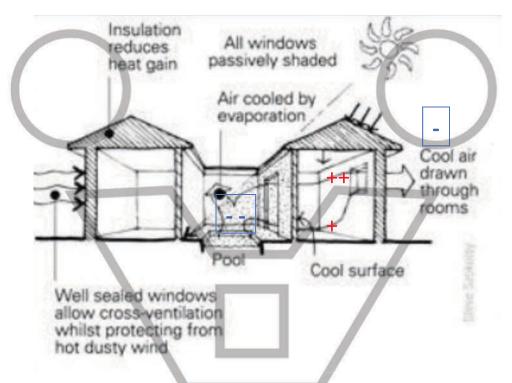
9.2

Airflow within a room is created by multiple sources air rising from (1) heat sources (such as a lamp, person, or a floor warmed by solar irradiation) due to buoyancy creating thermal stratification (red and blue overtones); the interactions of forced air from a (2) fan or a mechanical system and (3) outdoor air speed and direction, channeled by building geometry into and out of a room.

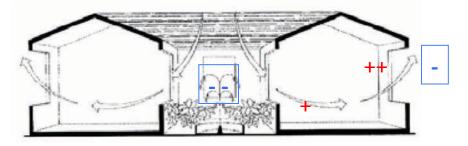
Air movement: VERTICAL VENTILATION



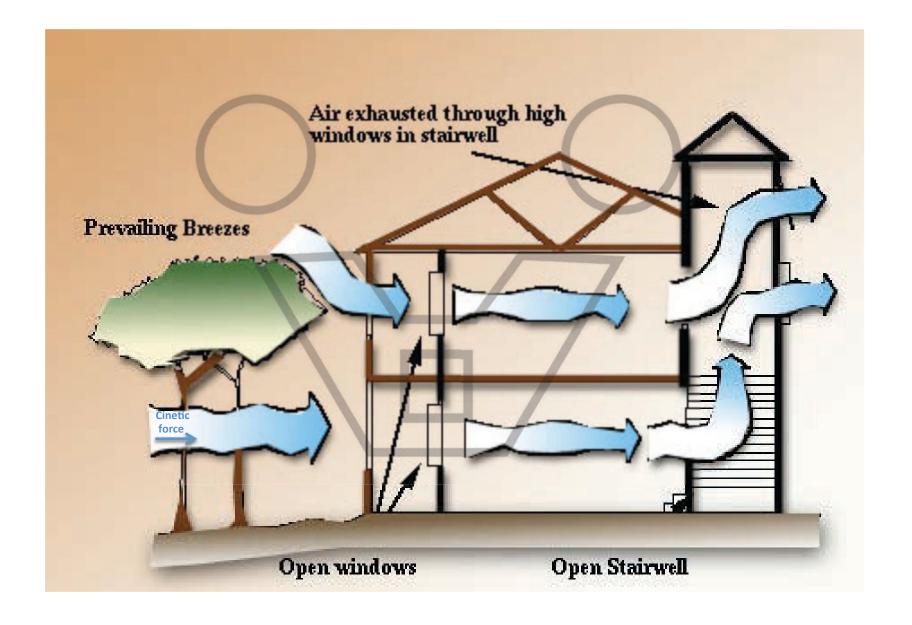
Air movement: VERTICAL VENTILATION



Low Pressure induced by cold zone needs to be carefully checked

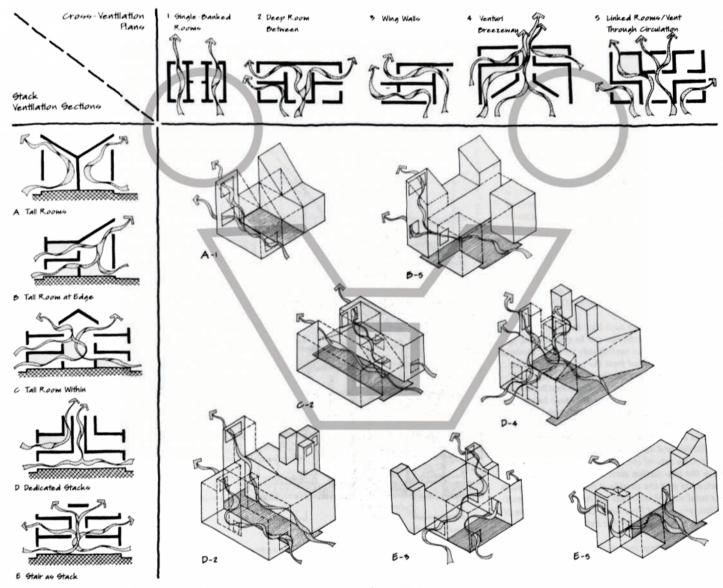


Air movement: VERTICAL VENTILATION



Air movement: VERTICAL VENTILATION

Working with natural ventilation Air movement: Cross ventilation + Stack effect room diagrams

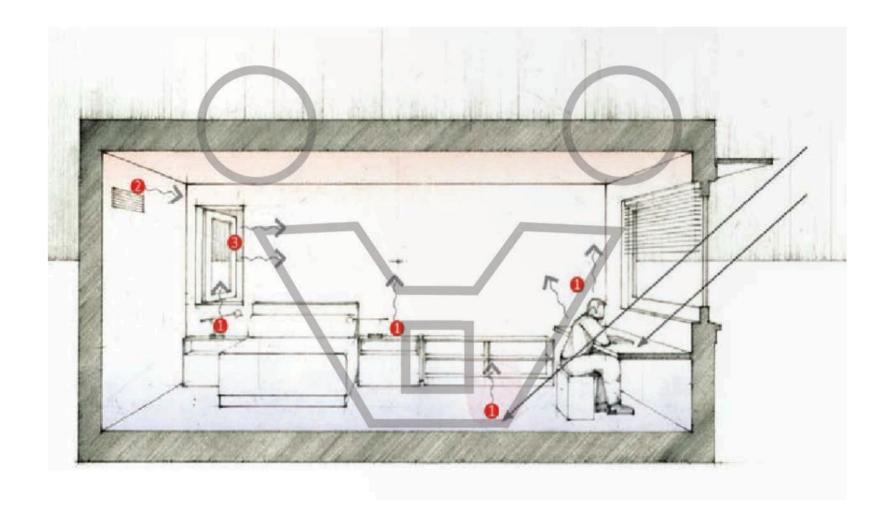


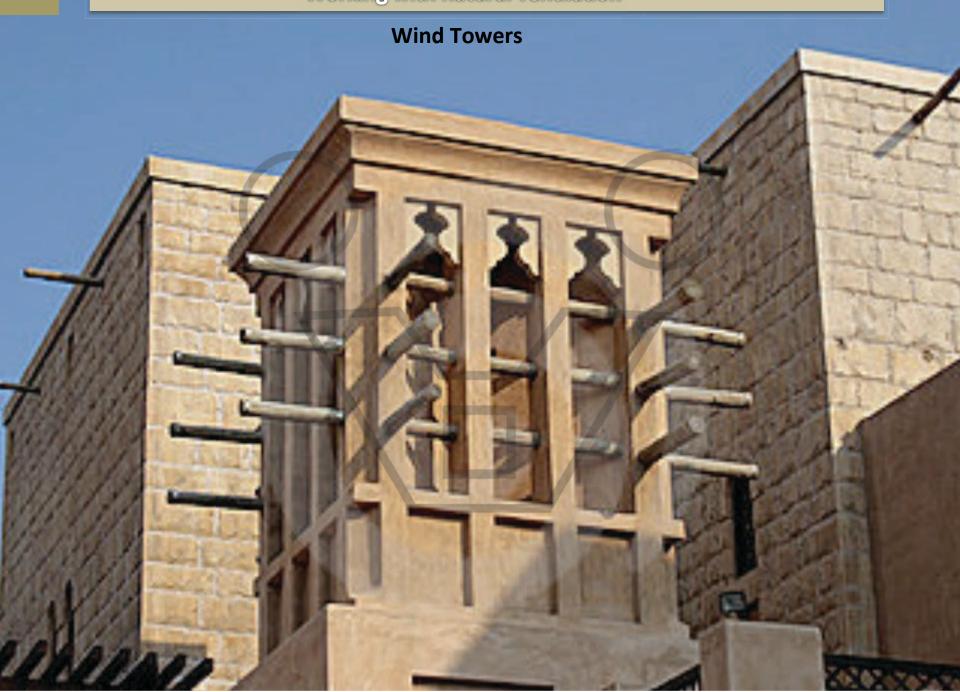
MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

WORKING WITH NATURAL VENTILATION

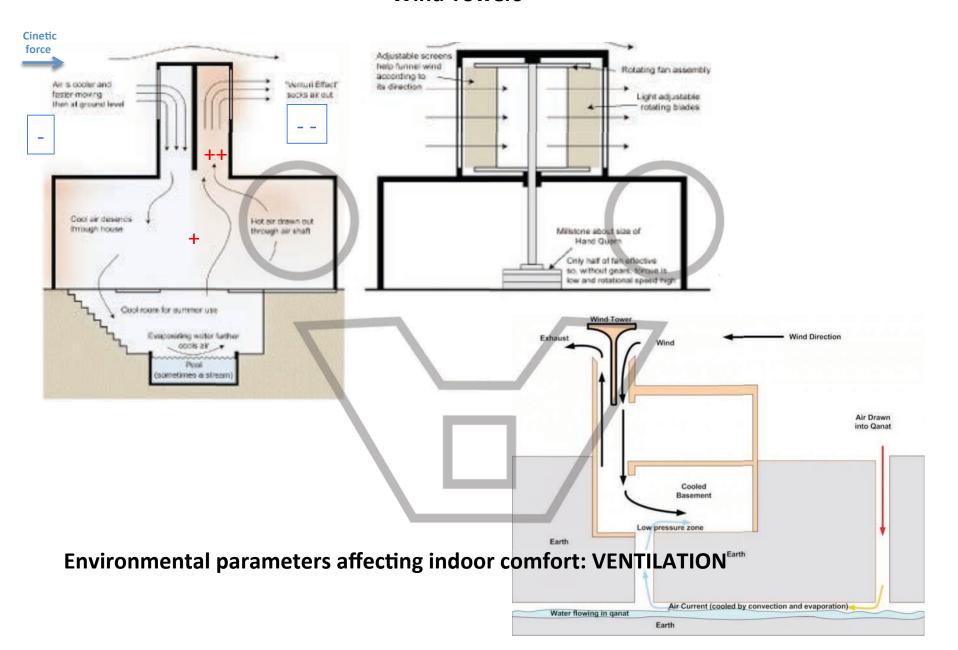
- To heat/cool through thermal convection
- to refresh through the sweating acceleration
- to clean exhausted indoor air
- to prevent condensation, moisture, and germs

Multiple sources airflow

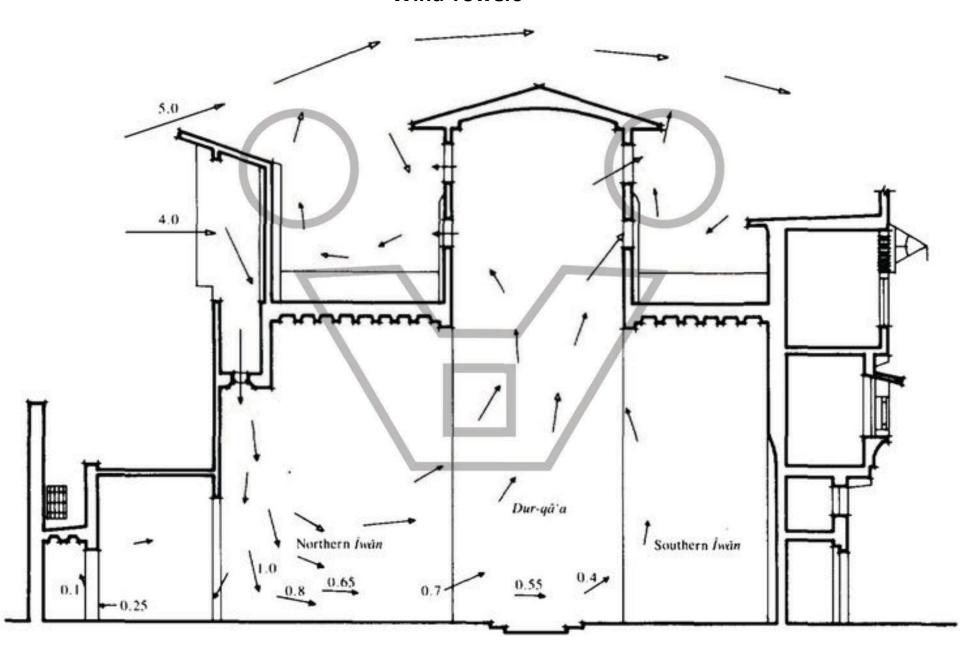




Wind Towers



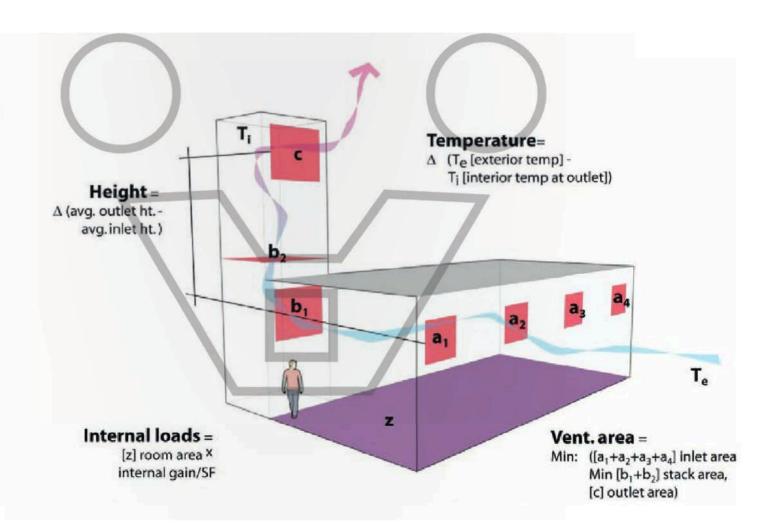
Wind Towers



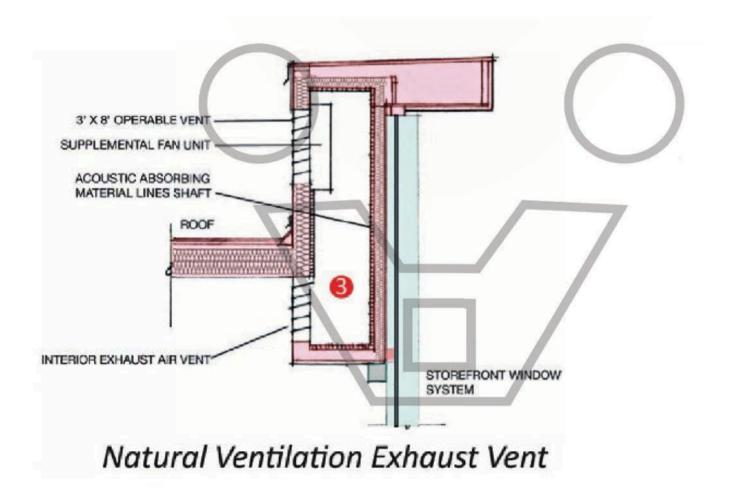
Wind Towers

9.12

Stack diagram showing the important inputs into a natural ventilation model.



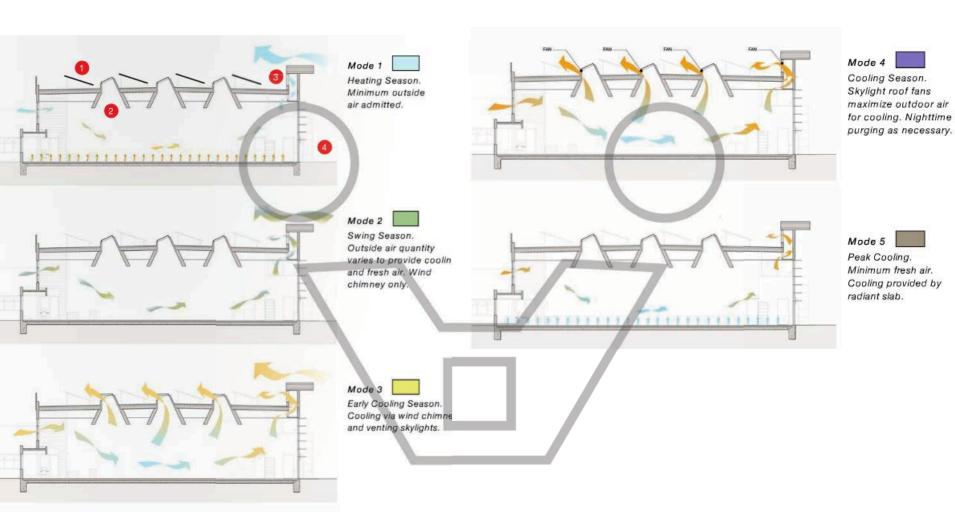
Wind Towers



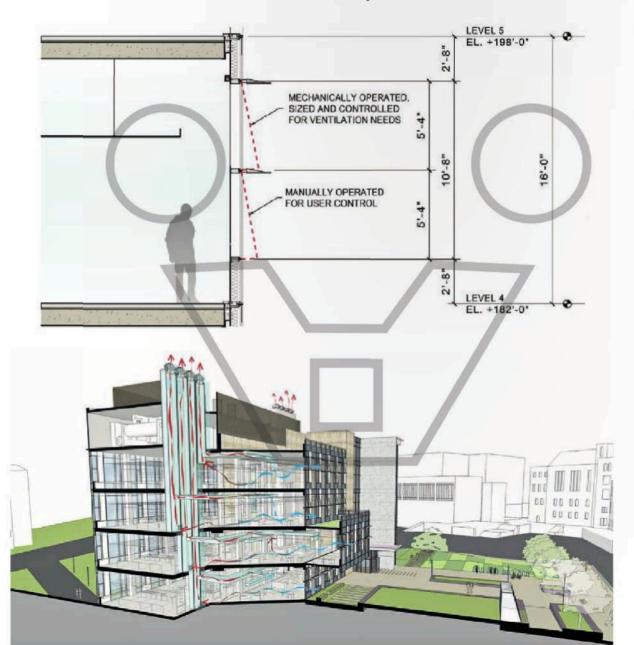
9.5

Natural ventilation exhaust vent.

Wind Chimney



Wind Chimney



9.13

Section through window showing window uses and sizes.

9.14

Natural ventilation diagram showing airflow into the offices and up through each floor's stacks.

Operable Windows



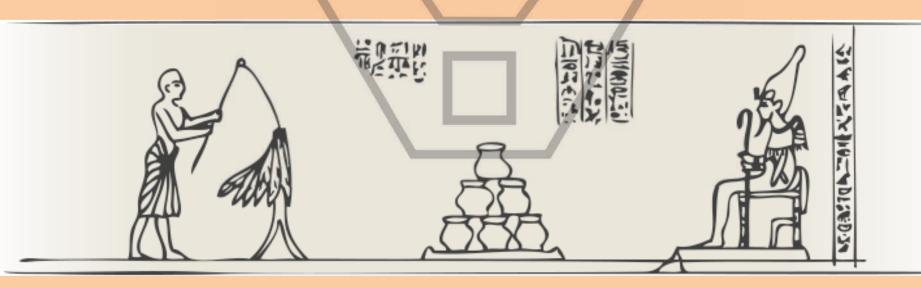
9.16

Photograph of the type of operable windows used at the Bullitt Center. Window diagram shows equal opening size around window's perimeter to reduce wear and provide even, controlled airflow.

Source: Photo and diagram courtesty Shuco.

MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

WORKING WITH WATER EVAPORATION IN HOT DRY CLIMATE







LATENT HEAT VS SENSIBLE HEAT

Latent heat is the energy absorbed by or released from a substance during a phase change from a gas to a liquid or a solid or vice versa. If a substance is changing from a solid to a liquid, for example, the substance needs to absorb energy from the surrounding environment in order to spread out the molecules into a larger, more fluid volume. If the substance is changing from something with lower density, like a gas, to a phase with higher density like a liquid, the substance gives off energy as the molecules come closer together and lose energy from motion and vibration.

Sensible is the energy required to change the temperature of a substance with no phase change. The temperature change can come from the absorption of sunlight by the soil or the air itself. Or it can come from contact with the warmer air caused by release of latent heat (by direct conduction). Energy moves through the atmosphere using both latent and sensible heat acting on the atmosphere to drive the movement of air molecules which create wind and vertical motions.

How much energy in water state transformation

0,09 W/h 0,079 kcal

Sostanza	Calore latente di fusione (J/g)	Temperatura di fusione (°C)		
Acqua	333,5	0		
Azoto	25,7	-210		
Alcol etilico	108	-114		
Ammoniaca	339	-75		
Mercurio	11	-39		
Zolfo	54	115		

0,63 W/gh 0,54 Kcal/g

1 litro = 630 W/h _____

1 litro = 45 W

Tempo medio 15-20'

— LATENT HEAT ABSORBED →

Liquid (water)

← LATENT HEAT RELEASED

Evaporation

Condensation

(water vapor)

Melting

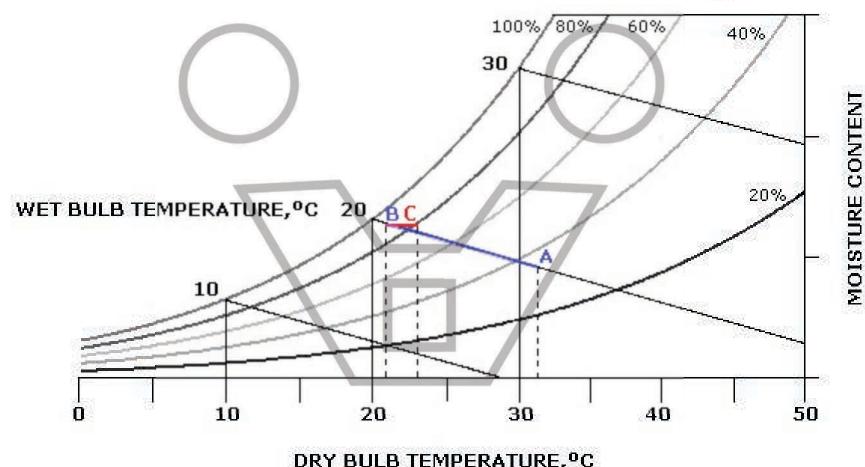
Freezing

(ice)

Sostanza	Calore latente di ebollizione (J/g)	Temperatura di ebollizione (°C)			
Acqua	2272	100			
Azoto	200	-196			
Alcol etilico	855	78,3			
Ammoniaca	1369	-33			
Mercurio	294	357			
Zolfo	1406	445			

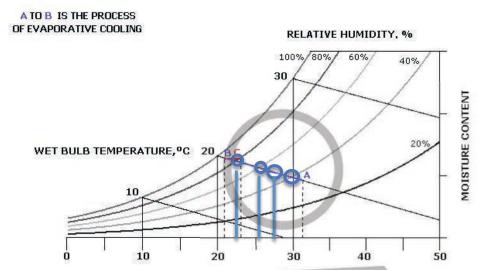
How much is the benefit from evaporative cooling





DRY BULB TEMPERATURE, °C

How much is the benefit from evaporative cooling



REAL TEMPERATURE:

30° (40%) >> 22°(80%) >> diff -8°

APPARENT TEMPERATURE

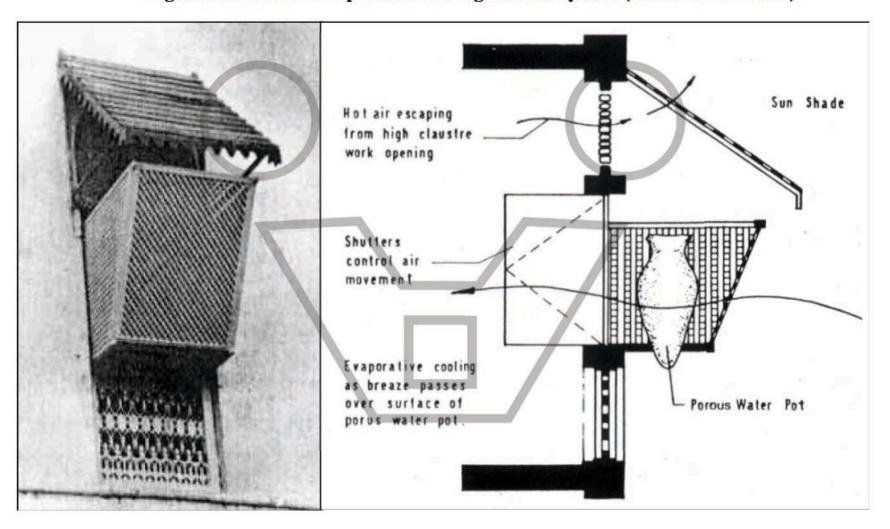
34° >> 30° >> diff -4°

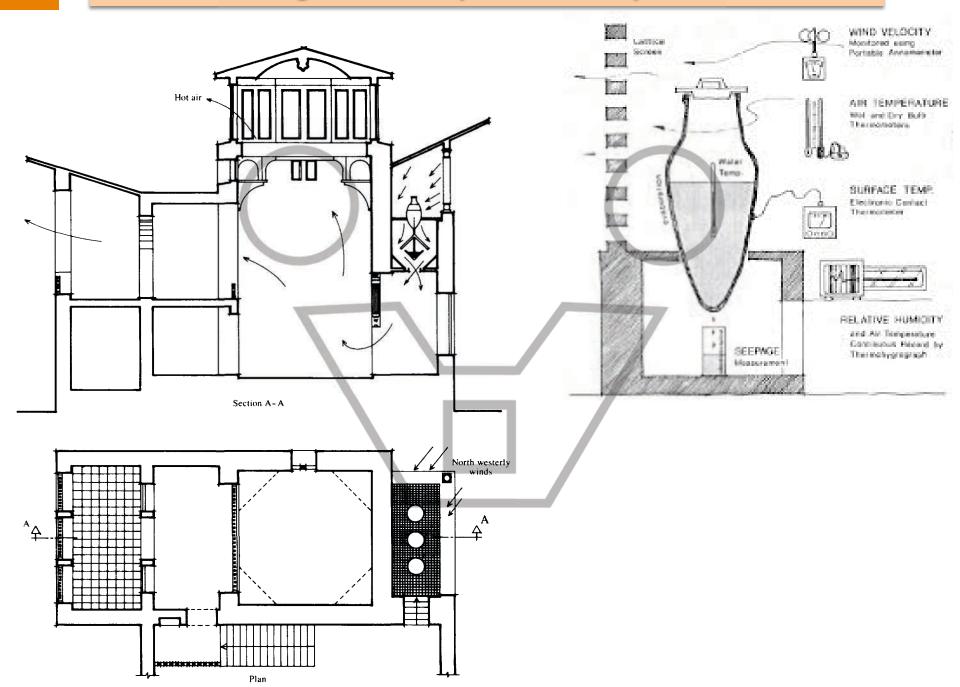
34° >> 28 >> diff -6°

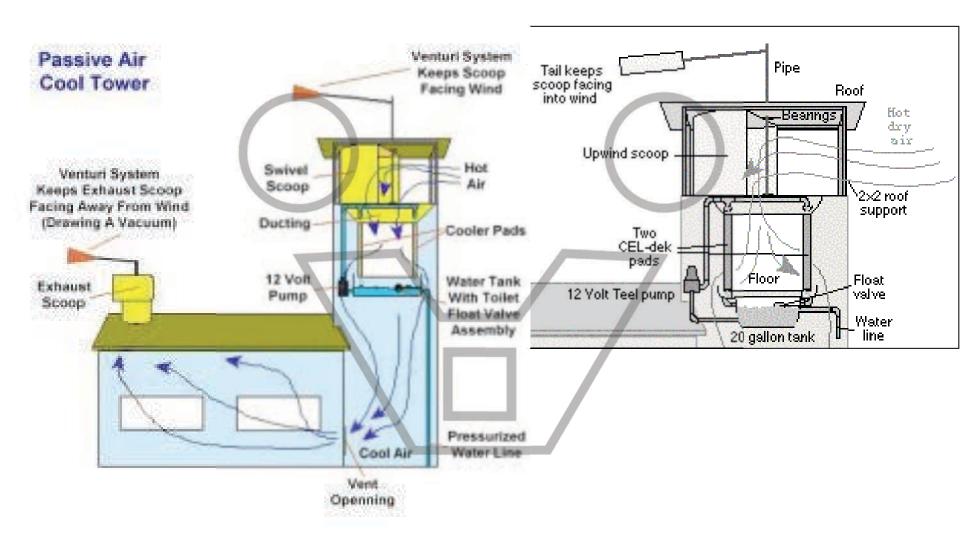
34° >> 32° >> diff -8°

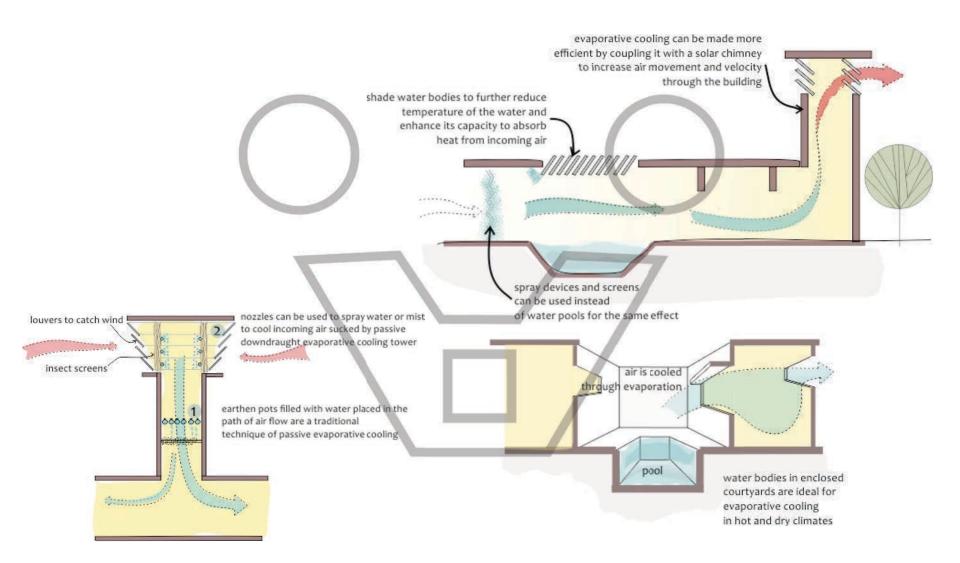
DRIVELUS TELLOSO LEURS DO															
DRY BULB TEMPERATURE, °C	25%	30%	35%	40%	45%	50%	55%	60%	65%	70%	75%	80%	85%	90%	959
42	• 48	50	52	.55	57	59	62	64	66	68	71	73	75	77	80
41	· 46	48	51	53	55	57 /	59	61	64	66	88	70	72	74	76
40	° 45	47	49	51	53	55	57	59	61	63	65	87	69	71	73
39	° 43	45	47	49	51	53	55	57	59	61	83	65	88	68	70
38	• 42	44	45	47	49	51	53	55	56	58	60	62	84	68	67
37	° 40	42	44	45	47	49	51	52	54	56	58	59	61	63	65
36	° 39	40	42	44	45	47	49	50	52	54	55	57	59	60	62
35	9 37	39	40	42	44	45	47	48	50	51	53	54	58	58	59
34	36	37	39	40	42	43	45	46	48	49	51	52	54	55	57
33	° 34	36	37	39	40	41	43	44	46	47	48	50	51	53	54
32	° 33	34	36	37	38	40	41	42	44	45	46	48	49	50	52
31	• 32	33	34	35	37	38	39	40	42	43	44	45	47	48	49
30	° 30	32	33	(34)	35	36	37	39	40	41	42	43	45	46	47
29	° 29	30	31	32	33	35	36	37	38	39	40	41	42	43	45
28	28	29	30	31	32	(33)	34	35	36	37	38	39	40	41	42
27	° 27	27	28	29	30	31	32	33	34	35	36	37	38	39	40
26	° 26	26	27	28	29	30	31	32	33	34	34	35	36	37	38
25	° 25	25	26	27	27	28	29	(30)	31	32	33	34	34	35	36
24	° 24	24	24	25	26	27	28	28	29	30	31	32	33	33	34
23	° 23	23	23	24	25	25	26	27	28	28	29	30	31	32	32
22	0 99	22	22	22	22	24	25	25	20	27	27	(20)	20	20	20

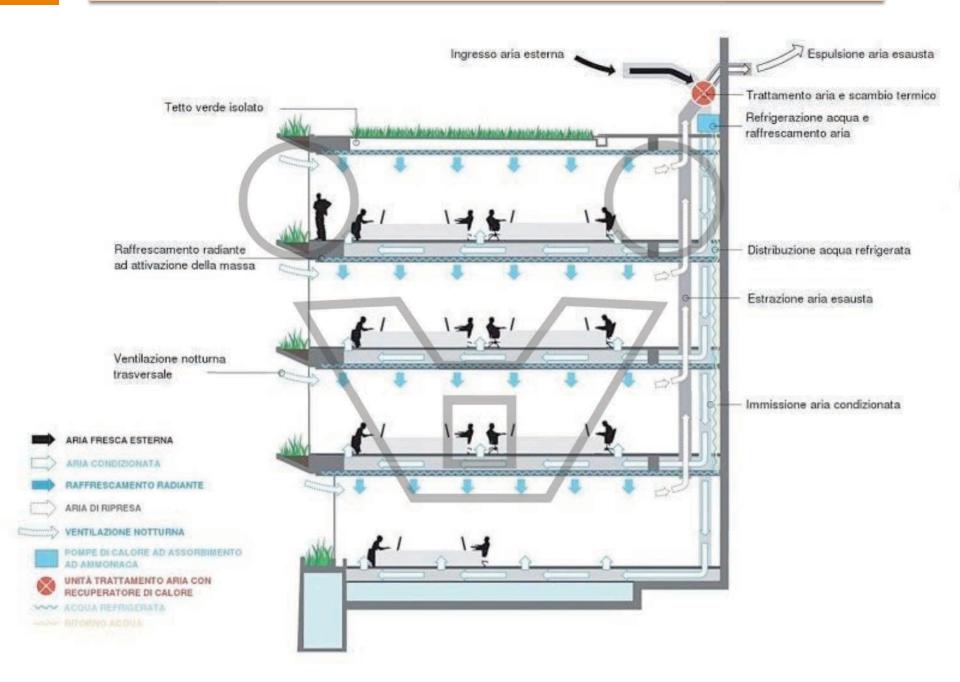
Figure: Muscatese Evaporative cooling window system (Rosa Schiano 2007)











MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

WORKING WITH MASS LATENCY or THERMAL LAG



What is THERMAL LAG?

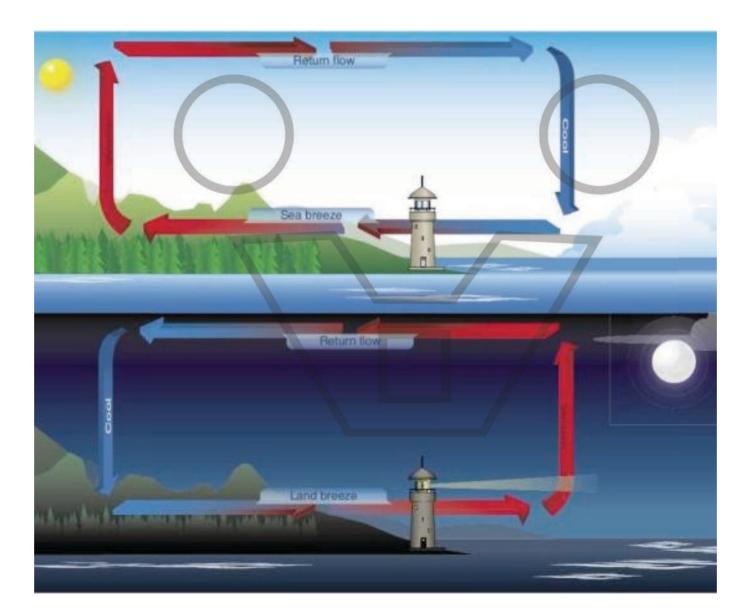
Thermal Lag describes a body's <u>thermal mass</u> with respect to time. A body with high thermal mass (high heat capacity and low <u>conductivity</u>) will have a large thermal lag.

Thermal diffusivity is the thermal conductivity divided by density and specific heat capacity at constant pressure

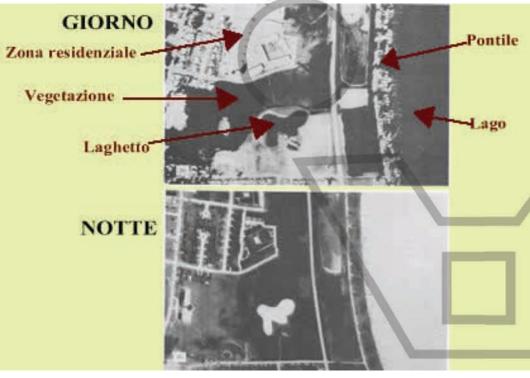
thermal mass is a property of the mass of a building which enables it to store heat, providing "inertia" against temperature fluctuations. It is sometimes known as the thermal flywheel effect.

This is distinct from a material's <u>insulative</u> value, which reduces a building's <u>thermal conductivity</u>, allowing it to be heated or cooled relatively separate from the outside,

A thermal flywheel effect from Nature: Marine breezes



Benefit of Thermal Mass



Thermal mass affects the temperature within a building by stabilising internal temperatures in three ways:

- stabilising internal temperatures by providing heat source and heat sink surfaces for radiative, conductive and convective heat exchange processes;
- *providing a time-lag* in the equalisation of external and internal temperatures; and
- providing a temperature reduction across an external wall (the decrement factor).

thermography

Internal temperatures stabilisation

Thermal mass influences comfort by radiant exchanges with the skin. In fact radiant exchange with mass surfaces is singularly the most efficient way of maintaining comfort compared with an other technique as the body is more that twice as sensitive to radiant losses and gains than all other pathways combined (conduction, convection, respiration, evaporation) and more than four times as sensitive than any other single pathway (see 2.3 below).

Thermal comfort exists when a body's heat loss equals its heat gain or *vice versa*.

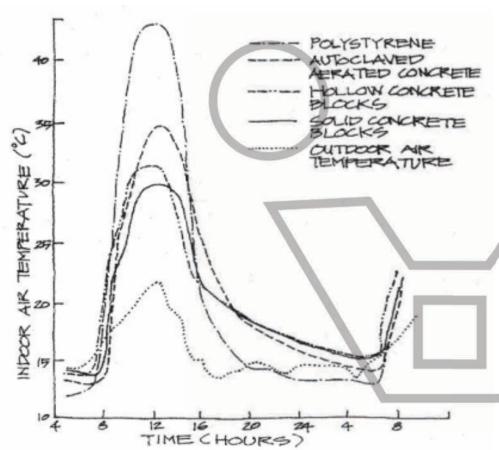
The body exchanges:

- ·62% of this heat via radiation,
- ·15% by evaporation,
- ·10% by convection,
- ·10% by respiration and
- ·3% by conduction.

http://www2.ecospecifier.org/

Relatively small changes in mean radiant temperature have a far greater effect than similar changes in air temperatures (Ballinger 1992). This gives rise to the importance of recognising the overall Environmental Temperature [T(env)], as opposed to just the dry bulb temperature.

Internal temperatures stabilisation

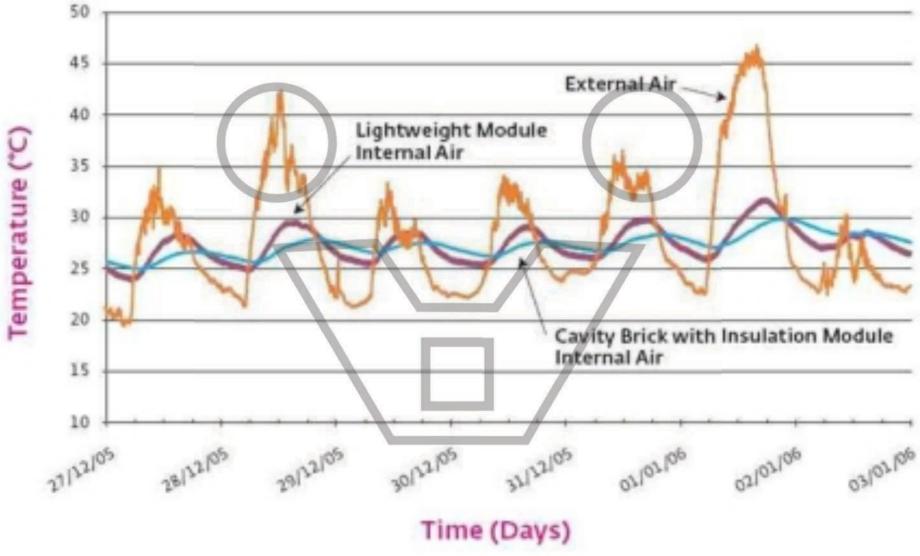


Thermal mass effects on diurnal indoor temperatures of various materials.

When heat enters a space directly by penetration of sunlight, lighting, equipment losses or heating, the temperature rise will be in inverse relationship to the accessible volume of thermal mass. Therefore, the indoor temperature will rise almost immediately if there is little thermal mass in the room. Figure uses an example of a simple box $1150 \times 1530 \times 1570$ mm, with a single window 660×1010 mm to demonstrate the effect of thermal mass on internal air temperature using a variety of materials.

This diagram represents unventilated spaces.

Internal temperatures stabilisation using different structural materials



Thermal mass effects on diurnal indoor temperatures of comparative insulated cavity brick & lightweight structures (Think Brick Australia 2006)

Heat capacity by materials

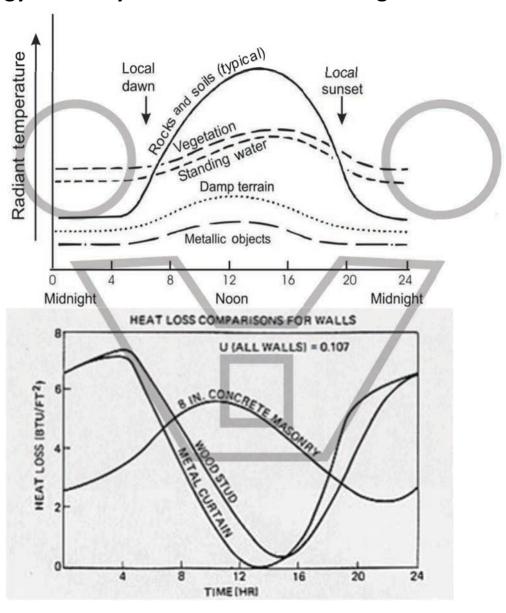
Specific heat is the amount of heat needed to raise the temperature of one kilogram of mass by 1 kelvin.

Material	Density	Specific heat	Volumetric heat capacity		
	(Kg/m3)	(kJ/kg.K)	Thermal mass (kJ/m3.K)		
Water	1000	4.186	4186		
Concrete	2240	0.920	2060		
AAC	500	1.100	550		
Brick	1700	0.920	1360		
Stone (Sandstone)	2000	0.900	1800		
FC Sheet (compressed)	1700	0.900	1530		
Earth Wall (Adobe)	1550	0.837	1300		
Rammed Earth	2000	0.837	1673		
Compressed Earth Blocks	2080	0.837	1740		

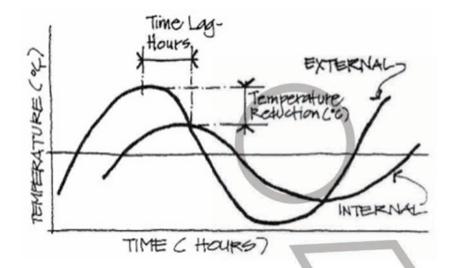
Table 1. Density, specific heat and thermal mass of a range of materials

Note: Figures are based on a number of sources and include estimations and interpolations. http://www2.ecospecifier.org/knowledge_base/technical_guides/thermal_mass_building_comfort_energy_efficiency

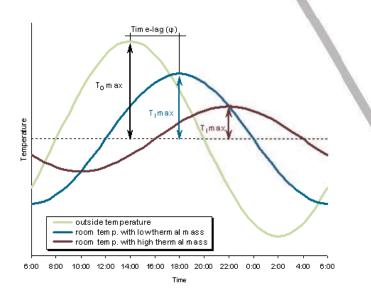
Radiant energy stored by different materials during the 24 hours



Time lag + temperature reduction



The effect of using heat generated during the day to warm at night in winter and vice versa in summer is known as the 'thermal flywheel' effect. The effectiveness of the flywheel depends on the time lag introduced to a building by an external wall or other boundary element. As can be seen from Figure 3, time 'lag' is the time delay between external maximum or minimum temperatures and internal maximum or minimum temperatures respectively

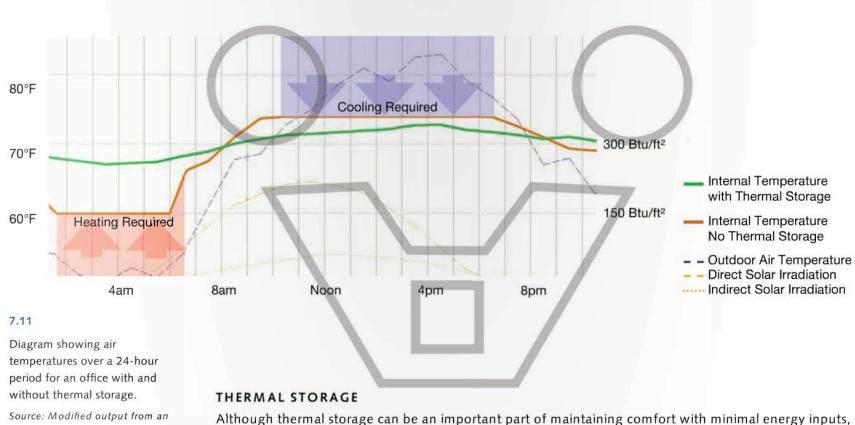


Material (thickness in mm)	Time lag (hours)
Insulated Brick Veneer	5.0
Concrete (250)	6.9
Double Brick (250)	7.0
AAC (200)	7.0
Adobe (250)	9.2
Rammed Earth (250)	10.3
Compressed Earth Blocks (250)	10.5
Sandy Loam (1000)	30 days

Table 4: Time lag figures for various materials (Baggs, SA, JC, DB., 1991) and (Think Brick Australia, 2006).

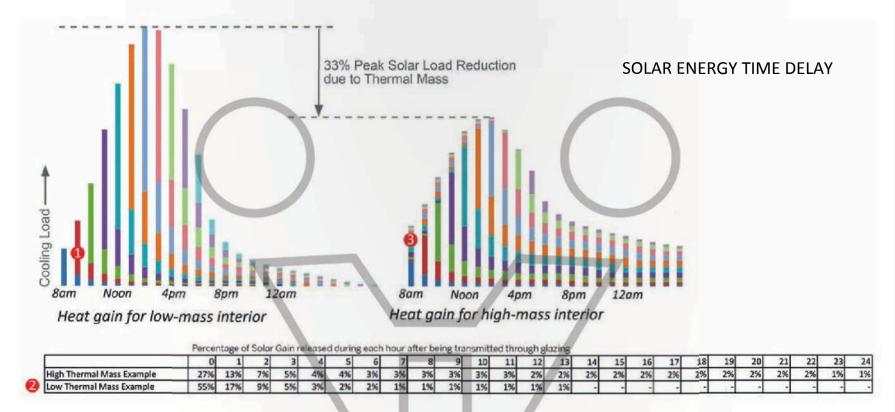
Effect of Thermal mass storage





Source: Modified output from an Autodesk Ecotect building model. Courtesy of Callison. Although thermal storage can be an important part of maintaining comfort with minimal energy inputs, over the past 200 years construction in much of the First World has tended towards lightweight, insulated buildings. Lightweight buildings are typically less able to use solar energy, since they cannot delay or

Effect of Thermal mass storage



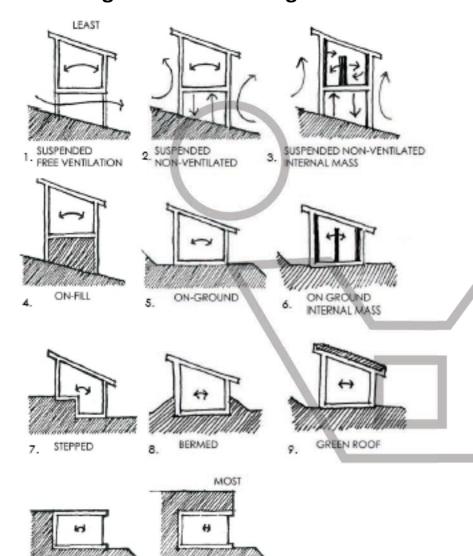
7.12

Solar irradiation values on a south-facing window in Toronto with a .50 glazing to wall ratio were imported onto a spreadsheet to calculate thermal mass effects on peak solar loading using the Radiant Time Series (RTS) method. Each hour's transmitted solar energy becomes a cooling load to the zone over the next 24 hours according to the percentages below for a low-mass and high-mass interior, which are color-coded to show the cumulative effects. At 9am, the solar irradiation that enters is colored red (1), and can be tracked over the next several hours until it becomes nearly negligible. For the low-mass option (2), 55% of the solar energy becomes a cooling load within the same hour it reaches the zone, and 27% is delayed until the second hour, with 9% becoming a cooling load in the third, etc. Each hour has been assigned a color to track it through the day, with the high-mass system including a small remaining solar load from the previous day (3) over the first several hours. The Radiant Time Series method (ASHRAE, 2013) is used to estimating peak cooling loads and contains an accurate but simplified version of estimating the time-delay of solar gain in low-, medium-, and high-mass constructions. The low-mass construction contains carpet, while the high-mass construction exposes concrete floors. The time-delay of other elements, such as exterior walls and solar energy absorbed by the glazing, was not considered. Solar irradiation values calculated in Autodesk Ecotect.

Source: Courtesy of Callison.

Locating mass in a building

10. WALL-CONTACT



11 EARTH-COVERED

HIGH IMPACT on CLIMATE DOMINATED BUILDING

- skinny buildings
- single houses,
- medium density residential,
- low-rise commercial buildings
- small scale educational and industrial buildings.

MEDIUM INTERNAL on LOAD DOMINATED BUILDING

medium and high-rise commercial and educational structures,

(Baverstock (1994) has shown that mass used in this way can provide 27% of the overall building cooling benefits and 38% of the overall building heating benefits.)

Locating mass in a building and operations in buildings with thermal mass

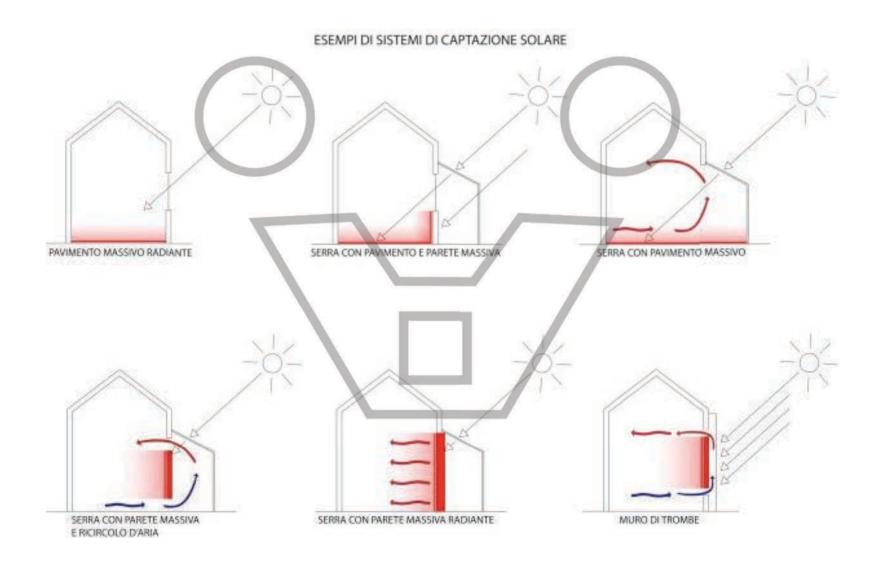
• External walls require minimum levels of added insulation for wall types under 200kg/m2

In the case of if adequate solar heat various kinds of earth walls such as adobe, rammed earth and compressed earth blocks, with their time lags of 10-11+ hours, is recommended left unsealed or finished with a 'breathable' paint.

Summer	Summer		Winter			
day	night	day	night			
closed	open	closed	closed			
closed	open	open	closed			
closed	open	open	closed			
	day closed closed	day night closed open closed open	day night day closed open closed closed open open			

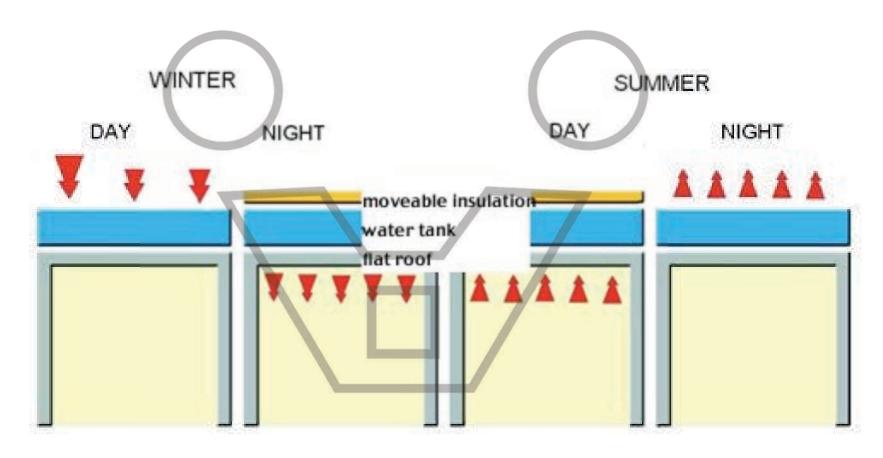
Table 2: User control of shading and ventilation devices

Locating mass in a building

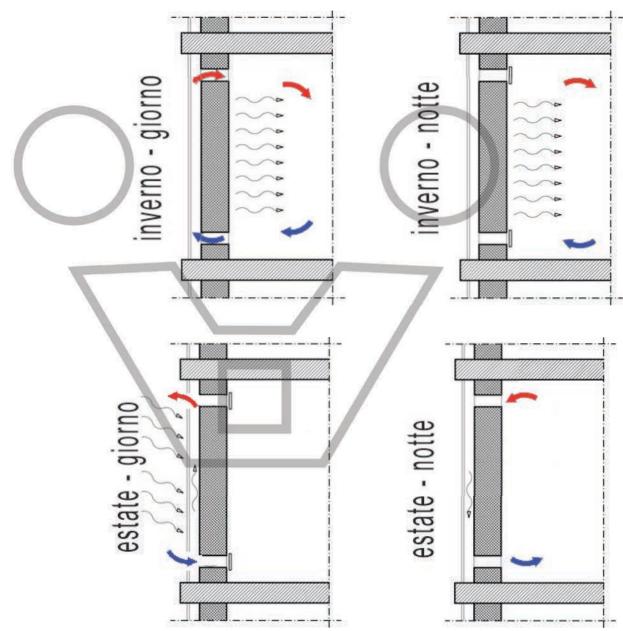


Locating mass in a building

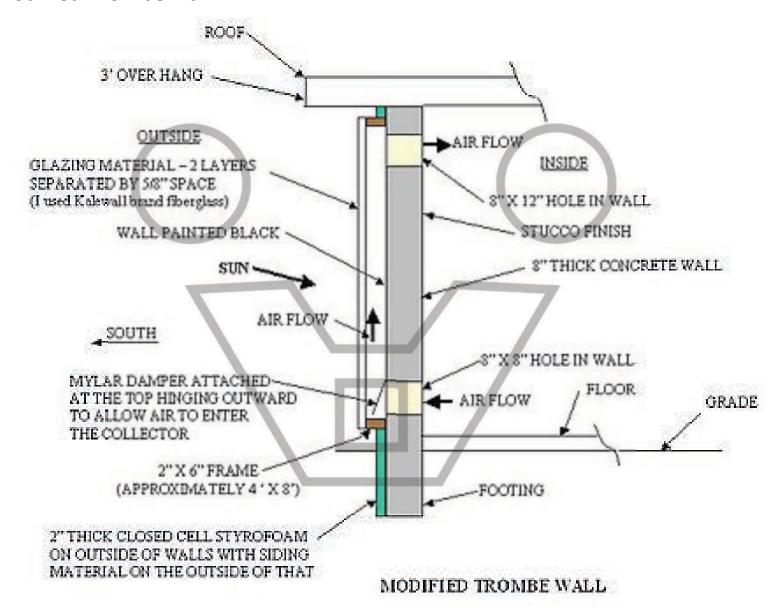
THERMAL STORAGE FOR HOT ARID CLIMATE



Trombe wall



Modified Trombe wall



Locating mass in a building

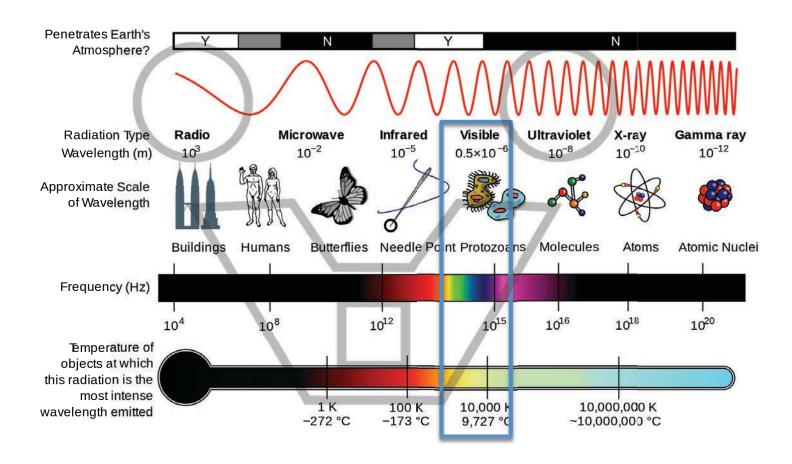


MODELING HUMAN BEHAVIOURS AND COMPUTING COMFORT CONDITIONS

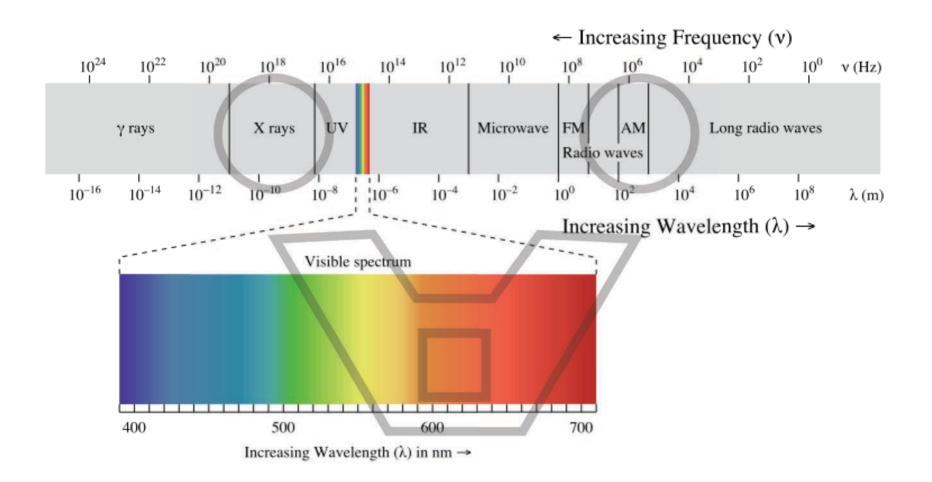
DAYLIGHT

Daylight vs thermal (and visual) comfort A conflictual relationship

VISIBLE RADIATION

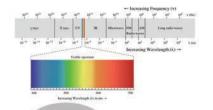


VISIBLE SPECTRUM

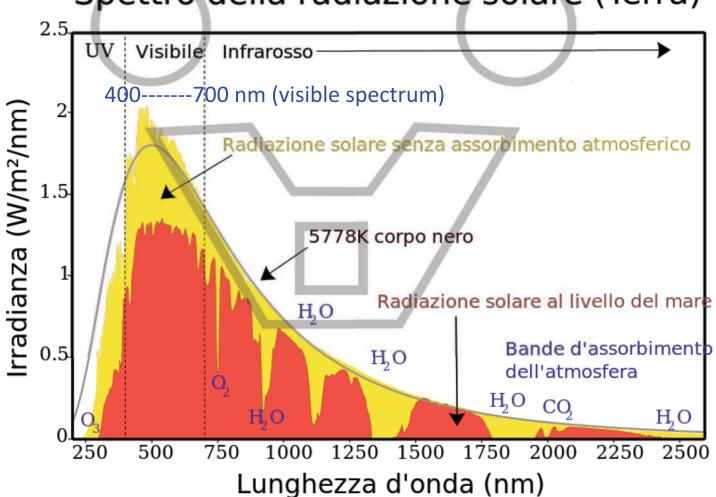


DAYLIGHT

ABSOLUTE ENERGY IN THE VISIBLE SPECTRUM

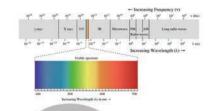


Spettro della radiazione solare (Terra)

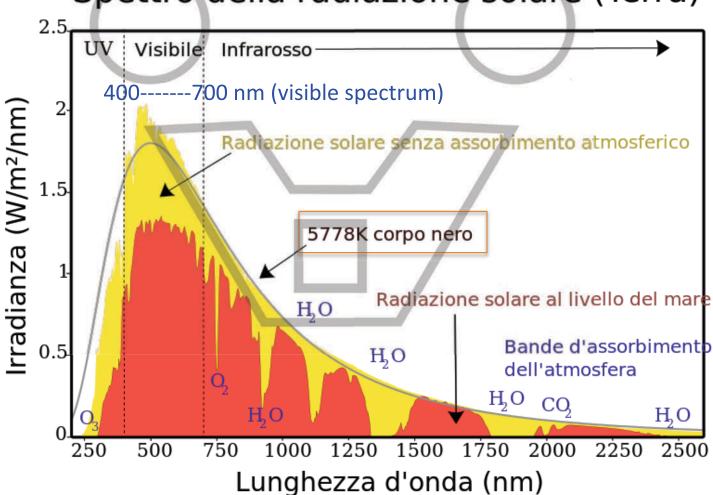


DAYLIGHT

ABSOLUTE ENERGY IN THE VISIBLE SPECTRUM



Spettro della radiazione solare (Terra)



RELATIVE ENERGY IN THE VISIBLE SPECTRUM

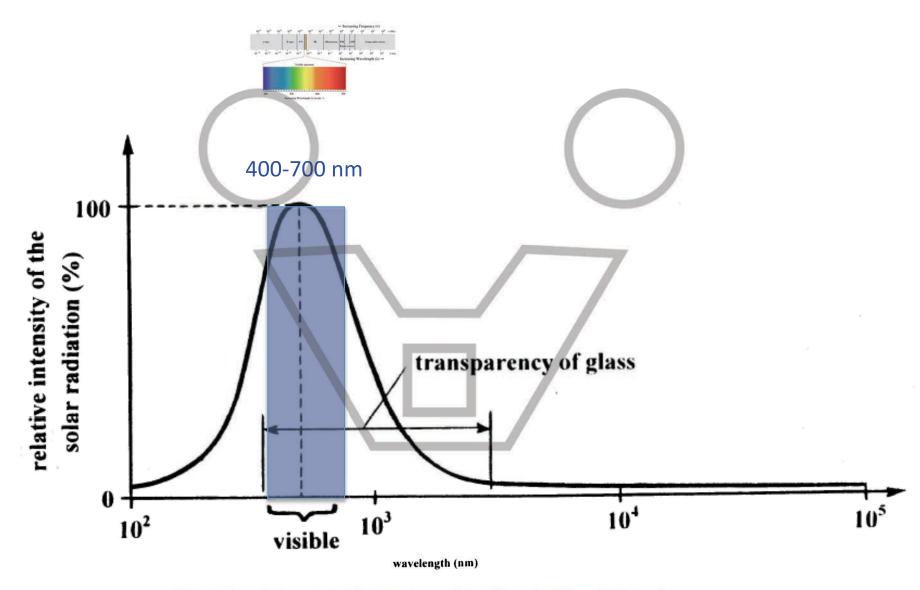
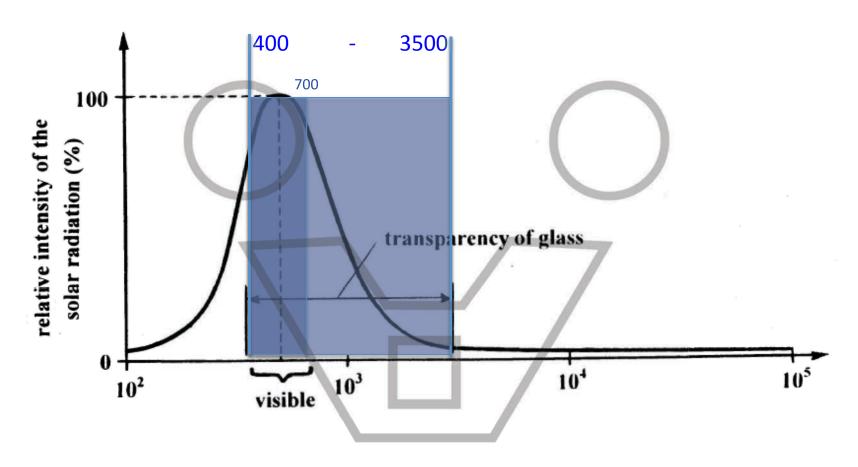


Fig. 7. Range di trasparenza del vetro rappresentato sullo spettro della radiazione solare. Un confronto con la figura precedente mostra che il vetro non è trasparente nel range di emissione dei corpi a temperatura ambiente o di poco superiore.

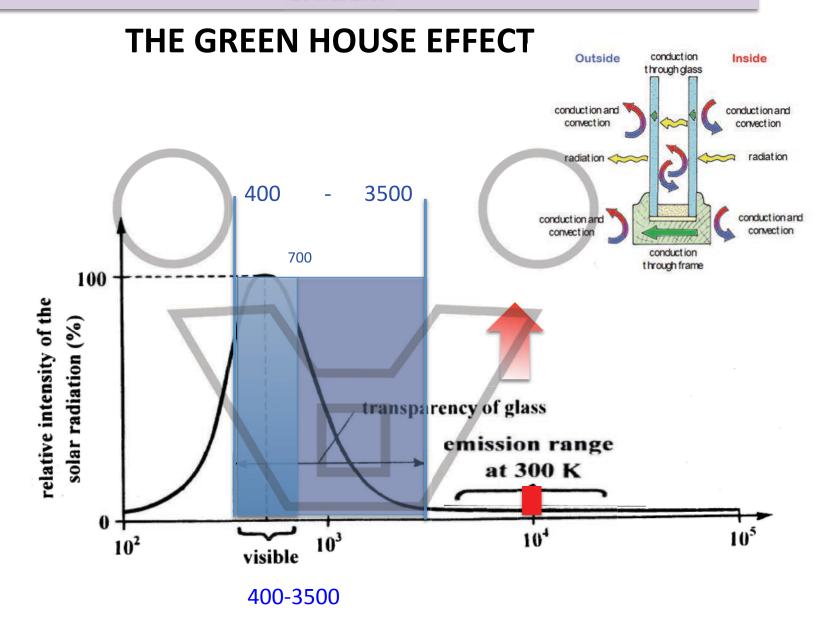
TRASPARENCY OF GLASS & % OF RADIATION PASSING THROUGH GLASS



wavelength (nm)

Fig. 6. Spettro della radiazione solare.

Sull'asse orizzontale è indicato il range di emissione relativo ad un corpo a 300 K. Lo spettro di emissione questo corpo non è rappresentato e sarebbe interamente sotto la curva dello spettro solare.



GLAZING MATERIALS AND GEOMETRY.

Spectrally selective coating

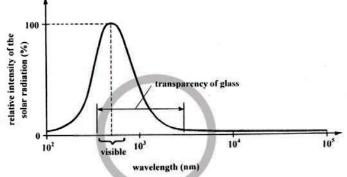
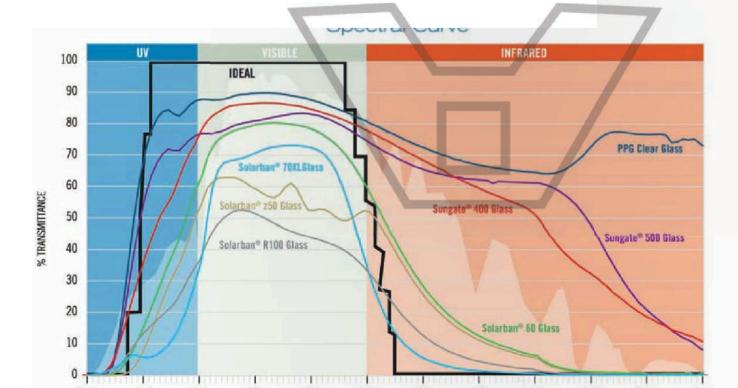


Fig. 7. Range di trasparenza del vetro rappresentato sullo spettro della radiazione solare. Un confronto con la figura precedente mostra che il vetro non è trasparente nel range di emissione dei corpi a temperatura ambiente o di poco superiore.

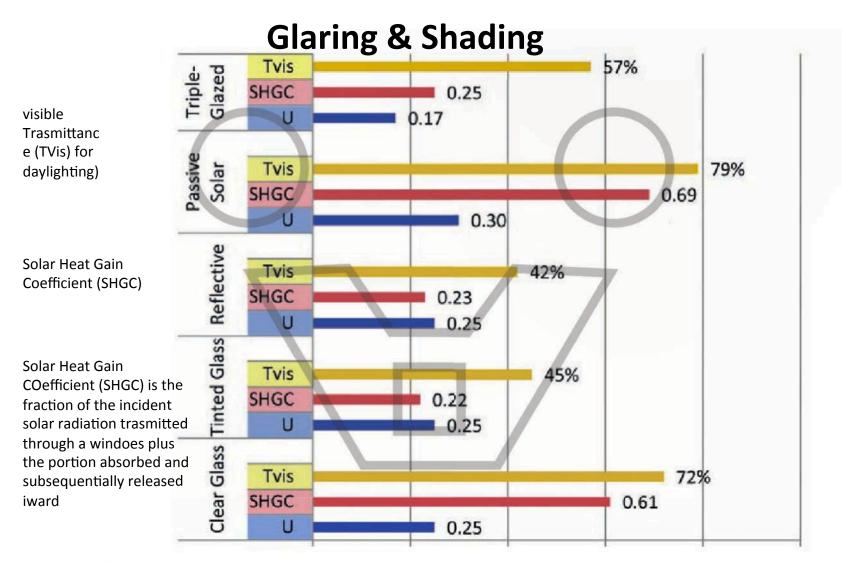


6.5

Spectrally selective coatings allow glazing products to reflect solar irradiation outside the visible spectrum without significantly reducing visible light transmittance. This allows low-SHGC products with high Tvis.

Source: Courtesy of PPG Industries.

Environmental parameters affecting daylight: GLAZING PROPERTIES

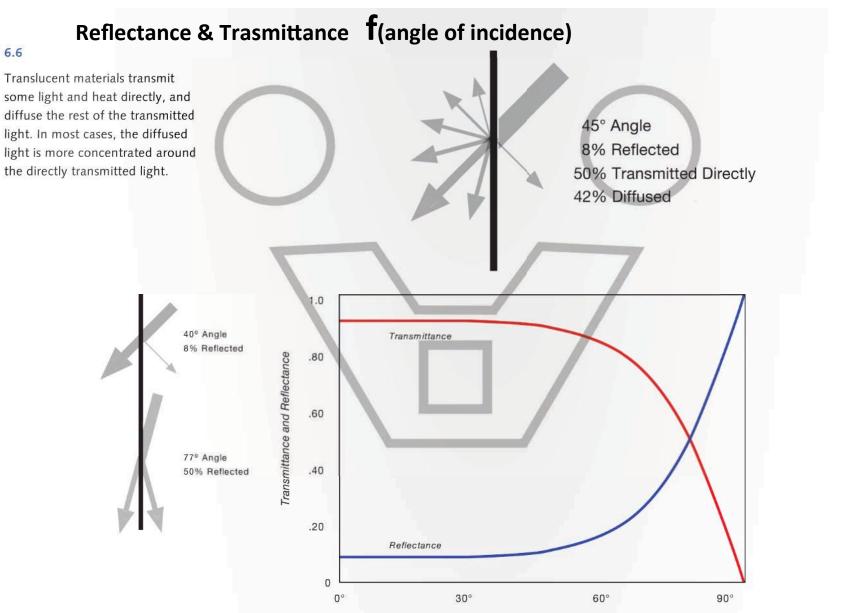


6.4

Glazing properties from some widely manufactured insulated glazing units (IGUs).

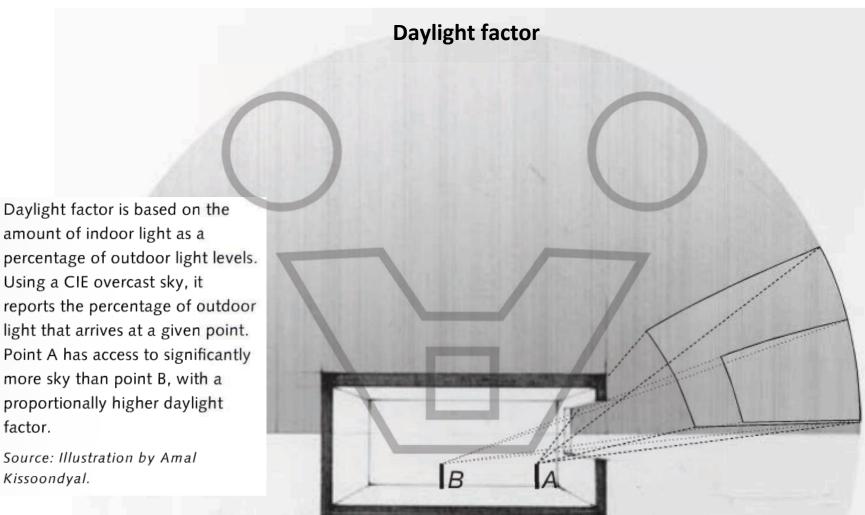
Source: Courtesy of Callison. Chart based on @ASHRAE Handbook of Fundamentals (2005), 31.20.

GLAZING MATERIALS AND GEOMETRY.



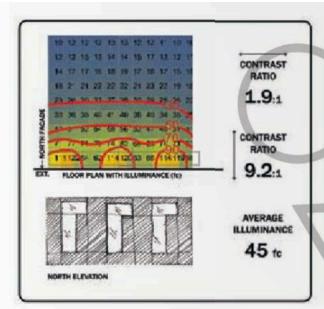
Angle of Solar Incidence on Window

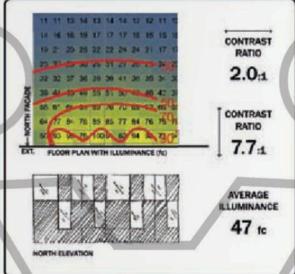
GLAZING GEOMETRY & MATERIALS

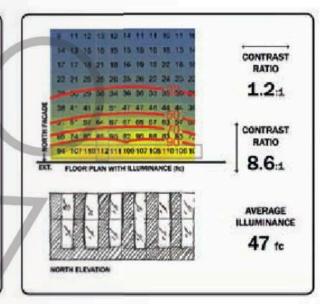


GLAZING GEOMETRY & MATERIALS

Illuminance levels







Plan-view studies showing illuminance levels for three window options on a north façade under overcast skies. The room's contrast ratios across width and depth are also shown, with lower contrast being preferable, but difficult to achieve, with side-lighting. Lighting designers typically include a room's contrast ratio in their studies to ensure even lighting throughout a space.

Source: Courtesy SERA Architects.

GLAZING GEOMETRY & MATERIALS

False Color Illuminance levels

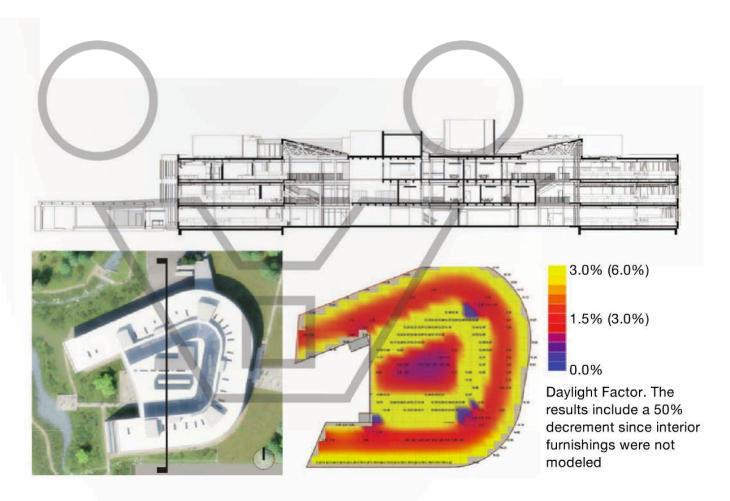


GLAZING GEOMETRY & MATERIALS

Day light false color analysis

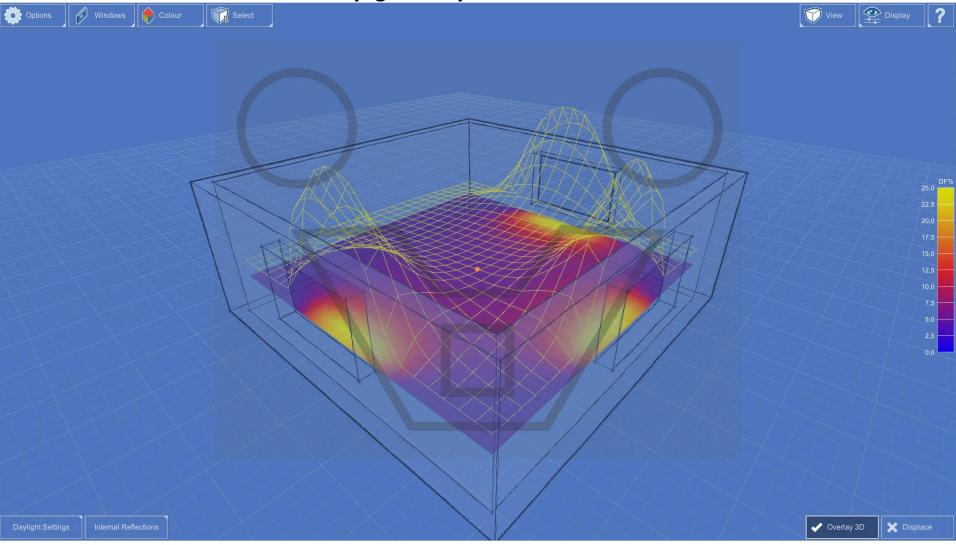
Federal Center South, Building 1202, is a 60'-wide office plate in the form of a U-shape around a daylit atrium. The false color results of a daylight factor simulation on the ground floor show good daylight based on sectional properties, including: office plate width, skylight geometry and glazing properties.

Source: Courtesy of ZGF Architects LLP.

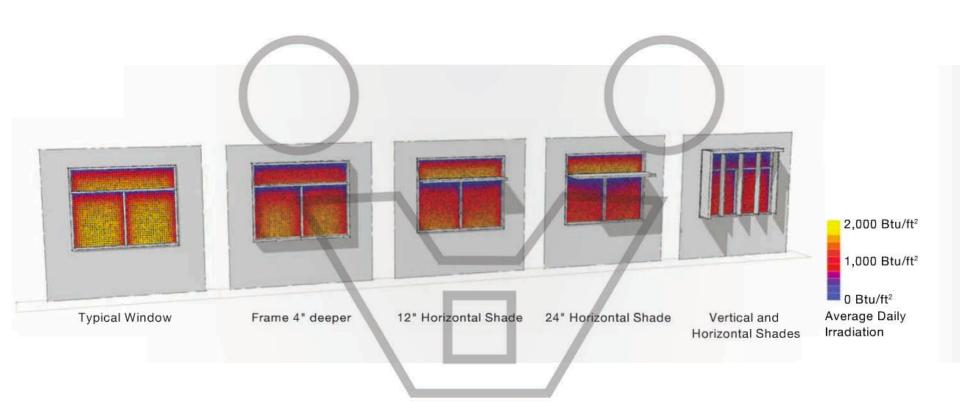


GLAZING GEOMETRY & MATERIALS

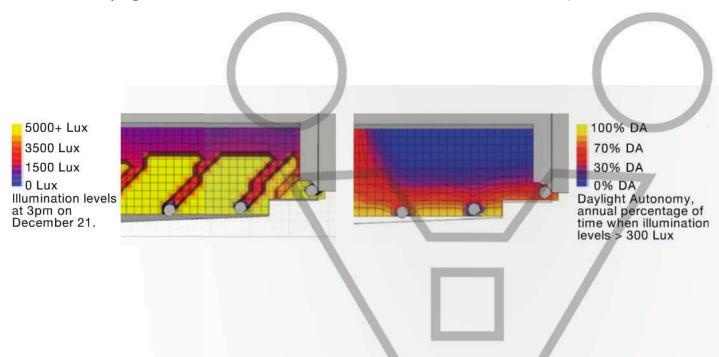
Daylight analyis



http://andrewmarsh.com/software/app-daylight/



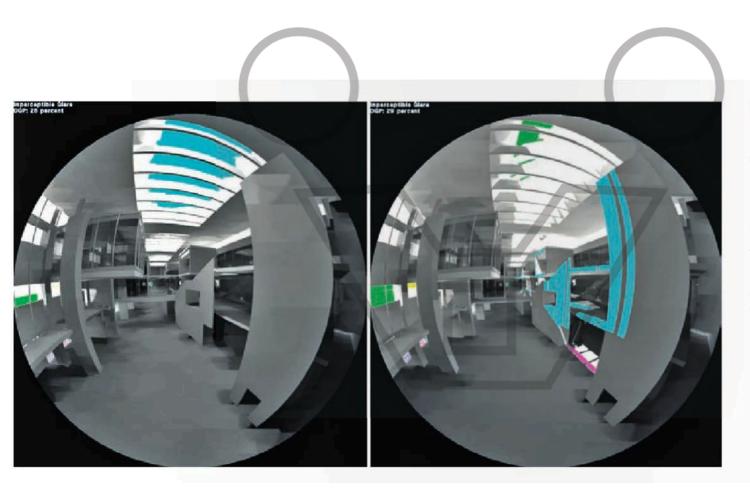
Daylight assessment: Point-in-time vs annual analysis



on investment of 3 years. If the project team assumes that occupants will turn off the lights when they leave at night, occupancy sensors may only be projected to reduce lighting energy use by 20%, with a return on investment of 10 years. As a broader example, ASHRAE 90.1's baseline building energy use is unique to every design, leading to industry-wide misunderstanding regarding a building's modeled

2.6

Plan view of an open office space, showing a façade with columns at the bottom. A point-in-time (PIT) analysis (left) provides information about daylight levels and potential for glare at 3:00 p.m. on the winter solstice, for example. An annual daylight autonomy analysis (right) shows areas that are successfully daylit for a certain percentage of the year. Both use false colors to illustrate lighting levels; they are Autodesk Ecotect outputs of Radiance and Daysim analyses, respectively.



8.12

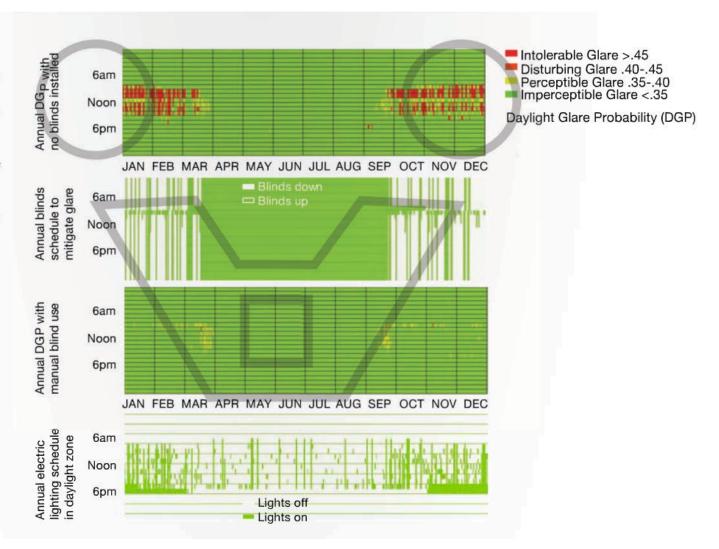
Daylight glare probability studies of atrium skylight options using DIVA software shows .26 and .29, both considered imperceptible glare. Each area within a field of view that contributes to glare is assigned a random color to show its location.

Source: Courtesy of SERA Architects.

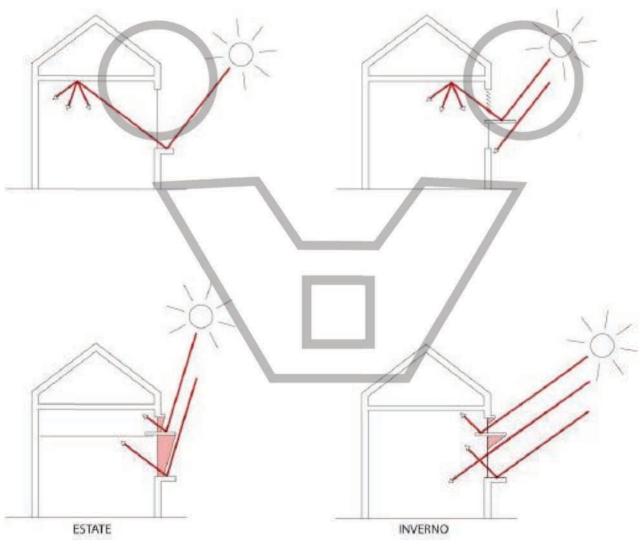
8.13

An east-facing viewpoint within a south-facing office space experiences glare primarily during times with low-angle sun in the Fall and Winter. Diva for Rhino software creates a blinds schedule to minimize glare, based on research of building occupants' tendency to lower them based on glare but raise them infrequently, per the Lightswitch model (Reinhart, 2002). The blind schedule helps create a lighting usage schedule that can estimate lighting energy use savings to compare design options.

Source: Courtesy of Jeff Niemasz.



Systems for daylight diffusion



Environmental parameters affecting daylight: WINDOWS GEOMETRY

Physical scale model to evaluate daylight and glare

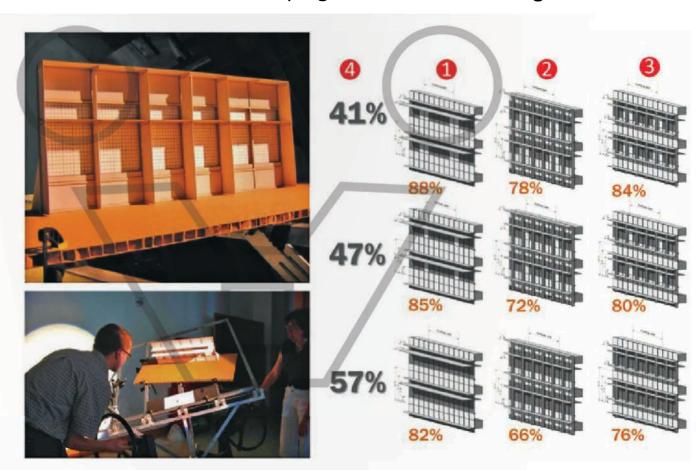




LIGHTING vs GLARING: Shaping windows and shading

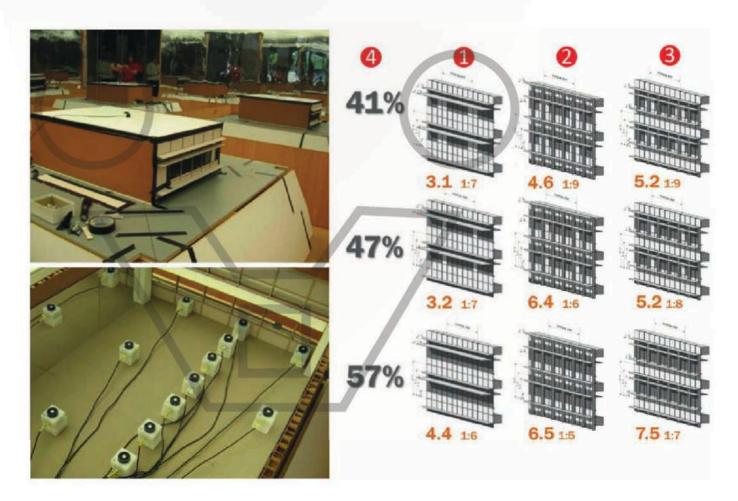
7.17

Façade shading studies testing options with (1) horizontal shade only, (2) vertical and horizontal fins with the horizontal element as a light shelf, and (3) vertical and horizontal fins with the horizontal as sill reflector. Each option was tested with 3 glazing percentage options (4). The orange number shows the percentage of the glazing that is shaded in each option.



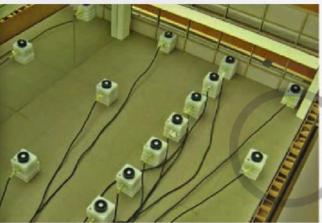
7.18

Daylighting studies testing the same 9 shading options as 7.17. The large orange number shows the average daylight factor in the 16' perimeter zone, and the small number shows the contrast ratio across this area.



Glaring & Shading









8.14

A physical daylighting model showing use of light meters to calculate the daylight factor. An overcast sky is simulated by the light box, which has highly reflective ceiling and walls to create uniform light levels.

Source: Courtesy of SERA Architects.

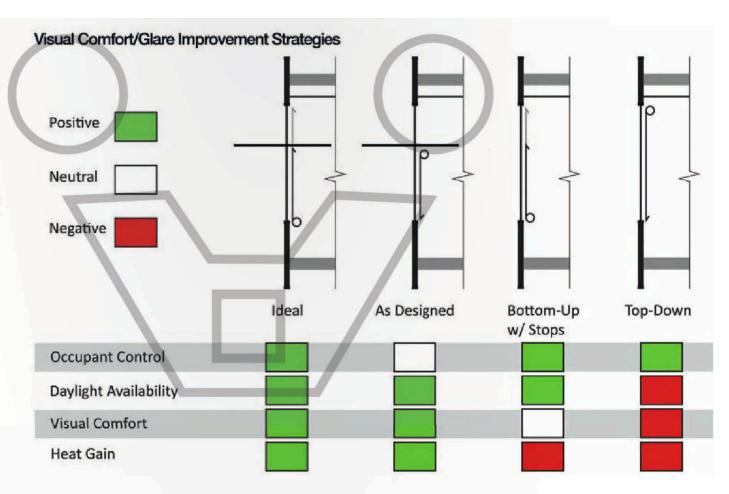
And a physical daylighting model showing use of a heliodon at the Energy Studies in Buildings Laboratory in Portland, Oregon, to predict daylighting levels under sunny sky conditions. The large wheels rotate the model through specific solar angles in relation to a bright electric light.

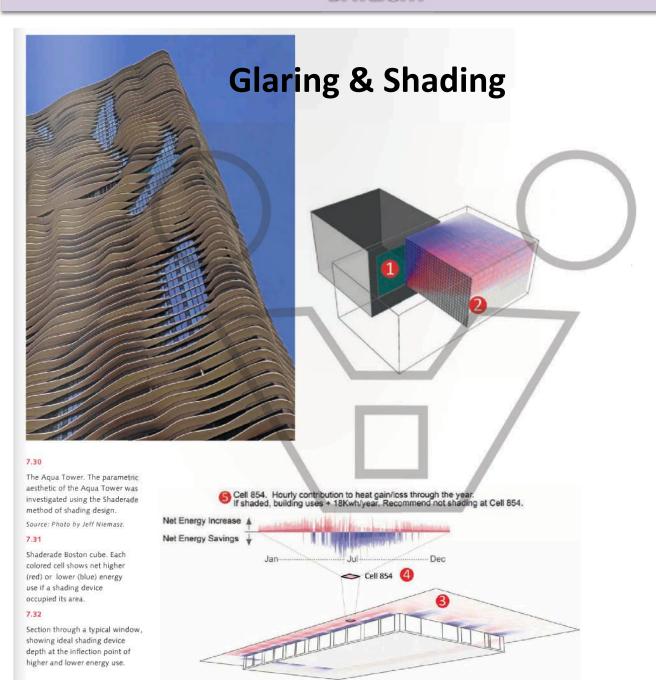
Source: Courtesy of SERA Architects.

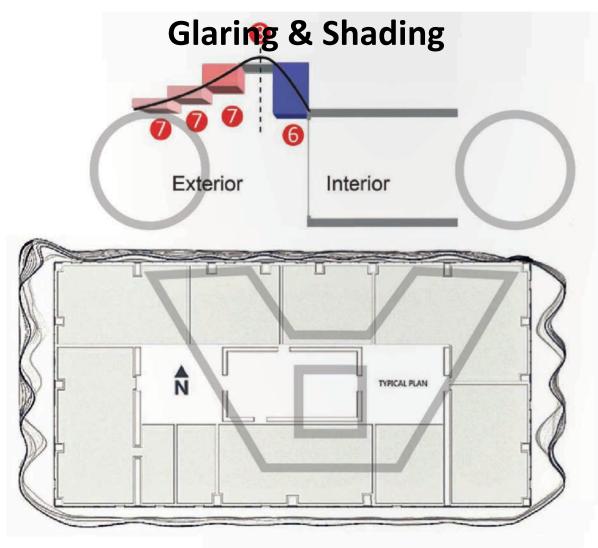
3.6

A study prepared for Iowa State University by ZGF Architects LLP rates four window options for user controllability, daylight availability, visual comfort, and heat gain. While simulations predict lighting energy savings due to the use of daylight, these savings are only realized when the system successfully blocks glare or allows users to block glare without blocking daylight.

Source: Courtesy of ZGF Architects LLP.







7.33

The inflection point.

7.34

Plan view of optimized shading.

The Shaderade approach, which advanced earlier work by Eran Kaftan and Dr. Andrew Marsh, involves mapping the annual energy consequences of each position where a shade could affect energy transmission through a window (1) in two or three dimensions (2).

The method begins by running a single simulation of a building shaded only by context, which

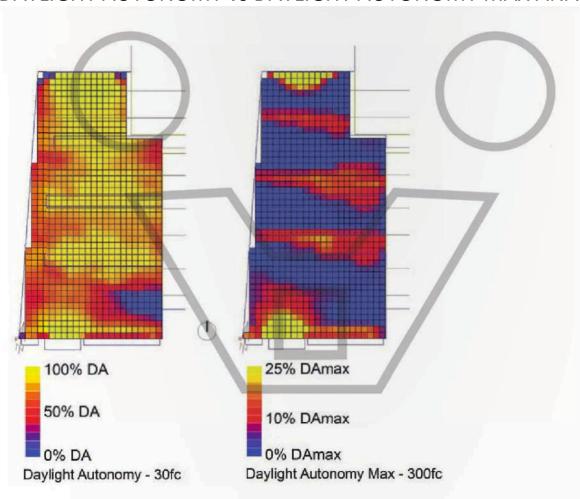
Zone Description	"zone"				
Occupancy Profile		User Requirements and Behav	vior————		
Select Occupancy Type	standard office	Minimum Illuminance Level	Minimum Illuminance Level 300		
Arrival Time	08.00	Occupant Behavior	Occupant Behavior		
Departure Time	17.00	Default behavior is active;	Default behavior is active; passive behavior tests 'design risk'.		
Lunch & Intermediate Breaks	P	Active Blind Control - User	Active Blind Control - User avoids discomfort glare (DGP >0.4).		
Daylight Savings Time	P				
Lighting and Shading Control System			0.0		
Installed Lighting Power Density	0.0	Standby Power			
Zone Size		Ballast Loss Factor	20		
Blind Control	No Movable Shading				
<u>Lighting Control</u>	Photosensor controlled	d dimming system	Specify Work Pl	ane	

8.10

Daysim software, that estimates daylight autonomy, has a user-inputted minimum lighting threshold, generally the illuminance level recommended for electric lighting. The characteristics of the occupants, the hours of operation, and other information are necessary to accurately estimate when electric lights may be dimmed or off on an annual basis.

A given sensor that reports a DA of 75% means that electric lights would not be necessary at the sensor during 75% of the occupied hours each year. DA presents a best case scenario for lights to be dimmed or off, since glare may cause blinds to be deployed, reducing light levels at the sensor. DA software requires the input of various assumptions about the building or users' operations of blinds as

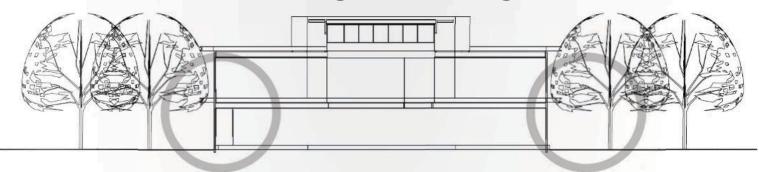
DAYLIGHT AUTONOMY vs DAYLIGHT AUTONOMY MAX ANALYSIS



8.24

Plan view of reading room showing annual DA and DAmax. The skylight geometry is shown to provide enough daylight throughout most of the year with over-lighting in only a few places for only around 10% of occupied hours.

Glaring & Shading GREEN SHADING MODELING



8.53

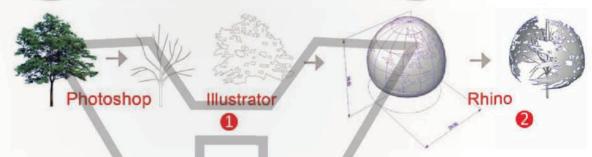
Sectional line drawing looking south, showing trees at two potential distances from the façade.

Courtesy of Skidmore, Ownings & Merrill, Chicago.

8.54

Creation of digital tree geometry.

Courtesy of Skidmore, Ownings & Merrill, Chicago.



Simulation

This simulation was performed by SOM's Design Performance Group, which specializes in early design simulations. The Revit-based model was imported and re-built in Rhino for this tree canopy study in the design development phase.

Research was conducted into modeling and defining the optics and seasonal behavior of a Thornless Honeylocust to create a simulated tree. A vector outline (1) of the species was projected onto two sides of a 3D volume. The canopy openness was approximated in the Rhino model (2) to allow direct, dappled light from various directions and a variation of light passing through and reflecting off leaves. This approach reduced meshed surfaces and simulation time.

Research and guidance from Christopher Meek at the University of Washington Integrated Design Lab (IDL) into the optical properties (3) of the leaves were used to create a Radiance material (4) for the leaves using Optics 6 Software. The tree leaves were scheduled to be present in the model between May 15th and October 15th to simulate deciduous vegetation.